

Operational Performance Analysis of the Cryogenic Electrical Machine for Submerged Liquefied Natural Gas Pumps

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The electrical machine is one of the key components for submerged liquefied natural gas (LNG) pumps. This study deals with a cryogenic permanent magnet synchronous motor (CPMSM) for submerged LNG pumps. An 18.5-kW CPMSM prototype was developed according to the needs of users. The basic design of the CPMSM was proposed by using a cryogenic temperature (CT) design method. To analyze the operational performance accurately, simulation modeling and analysis are carried out, and a liquid nitrogen (–196°C) experimental platform is built. The test results verify the rationality of the design and the accuracy of the simulation. Based on the simulation and experimental data, the working characteristics and mechanical properties of the prototype in liquid nitrogen are analyzed. In addition, the performance of the prototype operating in liquid nitrogen (–196°C) and LNG (–161°C) are compared and analyzed, and the results show that the operating characteristics of the prototype in the two fluid environments are similar.

Keywords: LNG pumps, cryogenic motor, finite element simulation, liquid nitrogen test, operational performance

1 INTRODUCTION

In recent years, with the large promotion and application of liquefied natural gas, LNG pump has a broad application prospect in the natural gas pipeline transportation, transport ship loading, unloading, and car filling (Tessarolo et al., 2011). With the advantage of safe, simple operation and some other strength, submerged LNG pump has become the main structural form of LNG pump (Rush and Hall, 2001; Zhu et al., 2012). However, the LNG pump technology is mastered only by United States' J. C. Carter Company, Japan's Ebara, Nikkiso, Shinko Company, France's Cryostar Company, and a few other companies. China does not have related independent intellectual property rights. Carrying out development related to submerged LNG pumps is very important (Hui et al., 2018).

A typical structure of submerged LNG pumps is shown in **Figure 1**. The drive motor is one of the core components of the pump. It is sealed in the pump body and immersed in LNG (-161°C). Foreign scholars have studied magnetic material selection (Liping Zheng et al., 2005; Dlugiewicz et al., 2006), electro-magnetic-thermal coupling simulation models (Shively, 2003; Kolowrotkiewicz et al., 2007; Baranski and Szelag, 2012), and operating characteristics (Kolowrotkiewicz et al., 2007; Ai et al., 2015). Zhu et al. (2012) and Guo et al. (2018a) pointed out that NdFeB is not suitable for the low-temperature environment below 140 K. In addition, the electromagnetic parameters and mechanical parameters of the motor under low temperature were simulated and analyzed. The

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operation efficiency was as high as 92%. However, the experiment was only carried out in a water-cooling test. Liping Zheng et al. (2005) studied the impact of liquid nitrogen (-196°C) on electrical testing equipment (material soaked in lowtemperature liquid to restore normal temperature), which cannot affect the magnetic properties of the permanent magnet material, and the remanence Br is slightly higher than the normal temperature. In the study by Kolowrotkiewicz et al. (2007), the test results show the starting torque reduction and the maximum torque increase with a high-pressure LNG pump in liquid nitrogen (-196°C) for the squirrel cage induction motor. Shively (2003) shows the asynchronous motor in liquid nitrogen (-196°C) is better than the motor at room temperature both in the working characteristics and mechanical properties. Since the stator and rotor of this kind of motor are immersed in cryogenic LNG during operation, excessive heat generation may cause LNG gasification and affect the operation efficiency of the pump system. At the same time, the application of the submerged pump requires its wide speed and variable power operation at a wide speed is required. Therefore, the pump motor is required to have the advantages of small heat output, high torque density, and high efficiency in a wide speed range. Ai et al. (2015) and Ai and Huang (2015), respectively, discuss the operational characteristics and key parameter design of cryogenic asynchronous motor for submerged LNG pumps. However, the asynchronous motor itself has certain limitations in efficiency and torque density, which is not conducive to the efficient and reliable operation with variable load in a wide speed range.

The cryogenic permanent magnet synchronous motor (CPMSM) is of high efficiency, high torque density, and large air gap, so it will gradually become the research and development trend of the cryogenic motor of LNG pump (Ai and Huang, 2015). The author of this study has conducted relevant research on CPMSM in the early stage (Guo et al., 2018a; Guo et al., 2018b). Since studies on the operational performance of cryogenic permanent magnet motor in the cryogenic environment (-161° C) have not been reported currently, based on the design of LNG pump cryogenic asynchronous motor (Miyagi et al., 2010; Pronto et al., 2011) and practical application experience, this study takes the 18.5 kW CPMSM prototype for submerged LNG pumps as the research

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Parameter	Value 18.5	
Output power P_N (kW)		
Pole pairs p	4	
Nominal voltage U_N (V)	380	
Nominal current I_N (A)	37	
Speed range n (rpm)	3,000-6,000	
Working temperature (°C)	-161	
Cooling medium	LNG	
Working type	Continuous duty	

object and carries out the simulation and experimental study on its efficiency, power factor, current, and torque under variable power. The research results will provide an important theoretical basis for the design, analysis, and operation control of the submerged LNG pump CPMSMs.

The rest of the article is organized as follows: Section 2 describes the design method of CPMSMs that is different from the conventional PM motors. The magnetic field distribution, electromotive force, voltage, and current of the proposed motor are simulated and analyzed in Section 3. The accuracy of the simulation model is verified by the prototype experiments in Section 4. The main operational characteristics of the motor are discussed in Section 5 based on the simulation and test results, and conclusions are drawn in Section 6.

2 DESIGN OF THE CRYOGENIC PERMANENT MAGNET SYNCHRONOUS MOTOR FOR SUBMERGED LIQUEFIED NATURAL GAS PUMPS

According to the type spectrum of TC34 soaking centrifugal pump in Nikkiso Clean Energy and Industrial Gases Group (www.cryopump.com), the power range of induction motor with a speed of 0–7,200 rpm at 380 V voltage level is 12.6–27. 1 kW. Referring to the product planning of similar cryogenic pumps manufactured by Dalian Deep Blue Pump Industry Co., Ltd., this study selects the motor with a power level of 18.5 kW and a maximum speed of 6,000 rpm as the research object. To meet the actual needs of users, an 18.5 kW CPMSM for a submerged LNG pump has been developed, and the specific parameters required are shown in **Table 1**.

2.1 Basic Design of the Cryogenic Permanent Magnet Synchronous Motor

The motor uses a built-in magnetic circuit structure. To increase the torque density and reduce the ends of the windings, V-shaped magnetic poles are adopted to place more permanent magnets. To reduce the rotational inertia and sufficiently cool, the rotor core is provided with axially symmetrical axial through holes in the circumferential direction.

Due to the decrease of the stator winding resistance and the increase of permanent magnet (PM) remanence in a cryogenic



environment, the conventional design method is not suitable for the proposed CPMSM. A cryogenic temperature (CT) design method is used to develop the CPMSM in this study, which suggests such a redesign idea based on an original conventional design: reduce the stator slot area by a certain ratio, which is the same as that of the conductor resistance decrease; also reduce the PM volume by a certain ratio, which is inversely proportional to that of the remanence increase. As shown in **Figure 2**, the slot area and PM volume to be redesigned by the CT method can be given by

$$\begin{cases} R = \rho_{Cu} \cdot \frac{L}{S}, R_{r,CON} = R_{cry,CT}, \\ S_{CT} = \frac{\rho_{Cucry}}{\rho_{Cur}} \cdot S_{CON}, \\ E_0 \propto V \cdot B_r, E_{0r,CON} = E_{0cry,CT}, \\ V_{CT} = \frac{B_{rr}}{B_{rcry}} \cdot V_{CON}, \end{cases}$$
(1)

where, R, $R_{r,CON}$, and $R_{crv,CT}$, respectively, denote the conductor resistance, the equivalent resistance at room temperature by conventional design, and the equivalent resistance at cryogenic temperature by CT design (Ω); L denotes the conductor length (mm); S, S_{CON}, and S_{CT}, respectively, denote the slot area, the slot area by conventional design, and the redesigned slot area by CT design (mm²); ρ_{Cucrv} and ρ_{Cur} , respectively, denote the conductor resistivities at cryogenic temperature and the conductor resistivities at room temperature ($\Omega \cdot mm^2/m$); E_0 , $E_{0r,CON}$, and $E_{0crv,CT}$, respectively, denote the no-load back EMF, the no-load back EMF at room temperature by conventional design and the no-load back EMF at cryogenic temperature by CT design (V); V, $V_{\rm CON}$, and $V_{\rm CT}$ respectively denote the PM volume, the PM volume by conventional design, and the PM volume by CT design (mm^3) ; B_r , B_{rr} , and B_{rcrv} respectively denote the PM remanence, the PM remanence at room temperature, and the PM remanence at cryogenic temperature (T).

The total volume of the motor can also be reduced by decreasing the motor diameter according to the reduction ratio of slot area and PM dimensions, while maintaining the slot pitch, pole-pair, air-gap length, and core length, as shown in **Figure 3**. It can be seen that by



FIGURE 3 Design of the cryogenic PMSM by the CT method compared with the conventional design.



using the CT method, the stator area can be reduced by nearly 1/3, and correspondingly the conductive material (copper) can be reduced by nearly 1/3; the rotor slot area can be reduced by 1/4, and correspondingly the permanent magnet area can also be reduced by nearly 1/4. However, the magnetic conductivity of silicon steel decreases at low temperatures. In order to keep the magnetic density unchanged, the reduction of stator and rotor area will be reduced. The outer diameter of the motor is reduced from 245 to 210 mm, which is nearly 1/5. According to the stator outer diameter standard, the motor size is reduced by a standard base center height.

2.2 Motor Efficiency

CPMSM in stable operation has the same loss as the motor at normal temperature, the electrical resistivity is 1/8 of that at room temperature, but the friction loss of the rotor and liquid nitrogen shall be considered, as shown in **Figure 4**.

Stator copper loss P_{Cu1} can be calculated according to the following formula:

$$P_{Cu1} = mI^2 R_1. aga{3}$$

Stator core loss can be analyzed according to the Bertotti iron loss calculation model (Miyagi et al., 2010), and the formula is shown as follows:



$$P_{Fe} = P_h + P_e + P_c = k_e f^2 B_m^2 + k_h f B_m^2 + k_c f^{1.5} B_m^{1.5}, \qquad (4)$$

where B_m is the flux density magnitude, f is the alternate frequency, k_e is the loss coefficient of turbine, k_h is the magnetic hysteresis loss, and k_c is the additional loss.

According to Dlugiewicz et al. (2012), the magnetic conductivity of silicon steel sheets is slightly improved at low temperature -196° C, but it is not different from that at normal temperature. However, the increase in iron loss at low temperatures is obvious when the flux density is saturated. Therefore, it is more appropriate to value the magnetic load of the motor at normal temperature in order not to increase the iron loss of the motor when designing the cryogenic motor.

The output shaft power P_2 of the motor can be obtained after deducting the mechanical loss p_{fw} and stray loss p_s from the electromagnetic power P_{em}

$$P_2 = P_{em} - p_{fw} - p_s, (5)$$

where the mechanical loss p_{fw} includes the bearing friction loss p_{fw2} and the friction loss p_{fw2} of rotor and liquid.

The bearing friction loss p_{fw1} of the motor at normal temperature is approximately equal to that of the motor at low temperature, and the friction loss p_{fw2} is several times that of the air at a normal temperature according to the density-dynamic viscosity ratio in the physical properties of liquid nitrogen shown in **Table 1**, so the friction loss p_{fw} of rotor and liquid cannot be neglected, and the calculation formula is as follows:

$$p_{fw2} = 0.5C_f \pi \rho \omega^3 L_{ef} \left(0.5D_{i2} \right)^4, \tag{6}$$

where ρ is the liquid density; L_{ef} is the core length of the motor [m]; ω is the rotation speed of the motor; D_{i2} is the rotor's outer diameter; C_f is the friction coefficient, which is related to the flow pattern and physical properties of the fluid in the air gap of the motor.

The stray loss p_s shall be calculated according to IEC 60034-2-1 (Spc, 2012), the stray loss at different loads shall be calculated according to that at rated load and the formula is shown as follows:

$$p_{\rm s} = \left(\frac{I_1}{I_N}\right)^2 P_{\rm sN},\tag{7}$$

where I_N is the rated current of the motor and P_{sN} is the stray loss when the motor outputs the rated power.

The calculated mechanical loss p_{fw} and miscellaneous p_s shall be substituted into **Eq. 5**, the output power P_2 of the motor can be obtained, and the operating efficiency of the prototype can be calculated according to the input power P_1 .

3 SIMULATION OF THE CRYOGENIC PERMANENT MAGNET SYNCHRONOUS MOTOR FOR SUBMERGED LIQUEFIED NATURAL GAS PUMPS

On the one hand, the motor shall be installed in the pump for the load test, so it is unable to directly measure the torque, speed, and other key parameters of the cryogenic motor; on the other hand, to ensure test safety, the motor shall be tested in liquid nitrogen (the temperature is -196° C under the pump pressure), the measured performance of the cryogenic motor has some differences from the actual operation in LNG (-161° C). Therefore, a more accurate simulation model of CPMSM is particularly important.

3.1 Simulation Model

The CPMSM is modeled in ANSYS software, as shown in **Figure 5**, based on the design mentioned earlier.

In view of the particularity of the application environment of the cryogenic motor, the distribution of the magnetic density and the friction coefficient of the rotor surface are very different from those of the common motor. In the simulation settings, the mesh generation is two times thinner than the ordinary generation to accurately analyze the magnetic density distribution of each surface, and the rotor friction coefficient is more than three times that of the conventional motor. The simulation time is set to 1 min to verify the stability and consistency of the output parameters.

Since the electrical resistivity ρ_{Cu} of the conductive material decreases, the residual magnetism *B*r of the permanent magnet increases, and the magnetic conductivity changes little at the cryogenic state, it is necessary to correct the equivalent circuit parameters according to the changes in ρ_{Cu} and *B*r parameters

TABLE 2 Simulation parameters of the prototype in liquid nitrogen (-196°C) and
LNG (–161°C).

Parameter	Liquid nitrogen	LNG	
Resistance R_1 (Ω)	0.0035475	0.004257	
Flux linkage φ (Wb)	0.50358	0.49427	
d-axis inductance L_{d} (mH)	0.99386	0.97025	
q-axis inductance L_{α} (mH)	2.1736	2.1219	
Rotational inertia J (kg.m ²)	0.01266	0.01266	

with the temperature when analyzing the cryogenic motor. It is also required to calculate the friction loss of the rotor due to liquid nitrogen $(-196^{\circ}C)$ accurately, and calculate the corresponding equivalent circuit parameters under $-196^{\circ}C$ liquid nitrogen, according to the parameters of the developed prototype, as shown in **Table 2**.

3.2 Finite Element Simulation of the Cryogenic Permanent Magnet Synchronous Motor

No-load and load finite element simulations have been carried out, respectively, according to the aforementioned method. **Figure 6** shows the magnetic flux density distribution of the motor, and the stator yoke and the tooth magnetic flux density are below 1.5 T under no-load conditions. The effective value of airgap flux density is about 0.5 T as can be calculated according to **Figure 6C**.

According to the simulation results shown in **Figure 6**, the stator and rotor magnetic density and air gap waveforms of the CPMSM are similar to those of the motor at room temperature under the condition of reducing the slot area and permanent magnet volume. There is no over saturation of magnetic density and distortion of magnetic field lines and air gap waveforms, which verifies the correctness of **Formulas 1**, **2**.

The no-load back EMF waveform of the motor is shown in **Figure 7**, and it can be known that the effective value of back EMF of line is 370 V from analysis after Fourier decomposition.

For safety reasons, the actual test will be carried out in a liquid nitrogen environment. Therefore, the parameters of conductive materials and permanent magnets used in the simulation are set according to the physical characteristics of the liquid nitrogen environment. The friction coefficient of the rotor surface in liquid nitrogen is completely different from that in air, so it should be set according to **Formula 6** during simulation.





For load simulation, it is usually set with reference to the actual experimental conditions. **Figure 8** shows the simulation waveforms of voltage and current in the liquid nitrogen $(-196^{\circ}C)$. From the simulation waveform and value of the load current and voltage of the CPMSM, it is within the range of design requirements. The waveform is smooth, and there is no distortion or three-phase current asymmetry. Due to the influence of spatial harmonics, the test waveform has a certain distortion, and the waveform obtained by simulation is relatively ideal, which often leads to some differences between the measured and the simulation results. The specific analysis and comparison will be explained in the experimental part of **Section 4**.

TABLE 3 | Characteristics of liquid nitrogen and LNG.

Parameter	Liquid nitrogen	LNG
Temperature θ (°C)	-196	-161
Density ρ (kg/m ³)	809	425
Kinematic viscosity μ (×10-6 Pa s)	165	120
Insulating	Fine	Fine

4 TEST OF THE CRYOGENIC PERMANENT MAGNET SYNCHRONOUS MOTOR FOR SUBMERGED LIQUEFIED NATURAL GAS PUMPS

As the test standards on the CPMSM are not seen yet in the documents, the study by Boglietti et al. (2000) and China's threephase permanent magnet synchronous motor test standard (Iec, 2014) are referenced. The load characteristics and the torque characteristics of the CPMSM prototype were tested, respectively.

As natural gas is flammable, the liquid nitrogen $(-196^{\circ}C)$ instead of LNG $(-161^{\circ}C)$ for the overall test of the pump is selected for safety reasons. The fluid properties of liquid nitrogen and LNG are shown in **Table 3**.

Due to the limitation to the operating environment, the power cannot be obtained by directly measuring the speed and torque of the cryogenic motor. Instead, the density and flow rate of liquid nitrogen will be determined by measuring the pressure and temperature of the inlet and outlet of the pump, and then we can find out the corresponding head and shaft power through characteristics of the curve of the pump impeller. At the same time, the motor input voltage, current, and other parameters can be measured with the oscilloscope's voltage, and current probe.

The test device is shown in **Figure 9**. The special controller for the permanent magnet motor GTAKE GK800 is used to drive the prototype. Since the test system is closed and self-circulating in the whole pipeline, it is necessary to add a bypass pipeline when testing the motor. After the motor is started, the valve is used to gradually switch to the liquid nitrogen pipeline, and then the selfcirculating test is realized.





When there is no load, the liquid nitrogen valve is turned off, the pump bypass channel is opened, and at this time, the readings of the export pressure gauge and flow meter are zero.

When there is load, the liquid nitrogen valve is opened and the output frequency of the controller is adjusted to improve speed. When the reading of the outlet pressure gauge reaches 2 MPa, readings of the flow meter greater than 20 m³/h (head 200 m), and the output power of the pump permanent magnet motor rated, the Tektronix MDO3014 oscilloscope is used to measure the waveform of the input line voltage U_1 and the input line currents i_1 , i_2 when the motor is rated, as shown in **Figure 10**.

Compared with the simulation waveform shown in **Figure 8**, the experimental results are basically consistent with that of the simulation. The corresponding simulation and test data are shown in **Table 4**. Compared with simulation and test waveforms, the current waveform recorded by the test is distorted to some extent due to the space harmonics, resulting in certain differences between the measured power factor and the simulation result. Overall, the simulation result shows high consistency with the test waveform, indicating that the established model of CPMSM can calculate the operating parameters of the motor accurately.

It can be seen from **Table 4** that the effective value of input voltage is 374.8 V at rated load and the effective value of the current is 53.5 A, which are measured when the pressure and flow in the pump are stable according to the readings of the pressure gauge and flow gauge. The test results show that the prototype can run well in liquid nitrogen, and the torque reaches the requirements of a submerged LNG pump, which proves the rationality of the design. However, it shall be noted that the tested current of the prototype (including the corresponding simulation results) is about 1.6 times the rated current of 37 A at the rated speed, as shown in the table. The reason is the density and viscosity of liquid nitrogen used in the experiment and simulation are both greater than those of LNG, so the output torque of the prototype in liquid nitrogen is higher, making the current larger.

It should be noted that the density of liquid nitrogen is 1.9 times that of LNG, and the kinetic viscosity is 1.35 times of LNG, that will result in greater output torque in the liquid nitrogen load operation at the same speed. Therefore, there must



be some differences between the test results and the actual operating results in LNG. **Figure 11** shows the power curve obtained when operating in these two media. At the same time, it is easy to conclude that if the prototype can run in liquid nitrogen for a long time, then it will reach the operating requirements of LNG.

5 ANALYSIS OF THE OPERATIONAL PERFORMANCE

Based on the aforementioned simulation and test data, the operational performance of the prototype can be further analyzed.

5.1 Operating Characteristics of the Cryogenic Permanent Magnet Synchronous Motor

According to the model and calculation program of the CPMSM established previously, the operating characteristics of the prototype under -196° C liquid nitrogen environment can be calculated, as shown in **Figure 12**. In the figure, the x-coordinate P_2/P_N is the ratio of output power P_2 and rated power P_N , and the y-coordinate is normalized, including the power factor $\cos\varphi$, efficiency η , and the per-unit value of stator input line current I_1^* .

The operating efficiency and power factor of the prototype are very high within the output power range of $0.2P_{\rm N}\sim 1.2P_{\rm N}$. The reason why the CPMSM has such excellent drive characteristics is that the fixed resistance of the motor at low temperature is small, the rotor has no loss, and the no-load of the motor is very small, so the efficiency curve increases rapidly with the increase in the load. When the variable loss is equal to the constant loss ($P_{\rm Cu1} + ps = P_{\rm Fe} + p_{\rm fw}$), the efficiency η reaches the maximum. As the output power P_2 continues to increase, the variable copper loss ($P_{\rm Cu1} = mI^2R_1$) of the CPMSM increases slowly and its efficiency slows down. Therefore, the CPMSM can maintain high efficiency and high power factor operation within a wide range of output power.

TABLE 4 | Simulation and experiment results of the prototype in liquid nitrogen (-196°C).

Parameter	Simulation value	Experimental value
Input voltage U (V)	408	412.3
Line voltage fundamental wave U_N (V)	370	374.8
Input current I _N (A)	55.4	53.5
Power factor $cos \varphi$	0.91	0.925
Speed ω (rad/s)	630	_
Torque T_N (N.m)	44	_
Pump output flow L (m ³ /h)	_	20.4
Pump discharge pressure p (MPa)	_	2.05







5.2 Mechanical Characteristics of the Cryogenic Permanent Magnet Synchronous Motor

The mechanical characteristics $T_{\rm e}$ -I of liquid nitrogen (-196°C) of the prototype can also be calculated, and the maximum electromagnetic torque $T_{\rm emax}$ can be obtained under different I-amplitude conditions.



The curve is shown in **Figure 13**, indicating the variation of the maximum electromagnetic torque of the same current output and the maximum torque value Kt ($Kt = T_{emax}/I$) of the corresponding unit current with the current space vector amplitude *I*.

Through the analysis of T_e^{-I} curve of the prototype, it can be found that the overall trend is basically the same as that of the motor at normal temperature, but there is also a certain difference. Owing to the increase in the residual magnetism Br of the permanent magnet at low temperature, the maximum torque is slightly greater than that of the motor at a normal temperature at the torque coefficient Kt in the linear region (the magnetic circuit is not saturated). However, the reluctance torque is rapidly decreased by the saturation of the magnetic circuit at low temperature in the linear region (the magnetic circuit is saturated), resulting in a decrease in the output torque of the prototype per unit, that is, Kt is smaller than that of the motor at normal temperature. Therefore, it is not recommended to increase the magnetic load of the cryogenic motor, which can be taken according to the normal temperature, that is, the flux density of each part of the motor rotor and stator does not exceed 1.5T (Miyamoto et al., 2011; Bernard et al., 2016; Lv et al., 2019), and the gap flux density Bg is about 0.5T.

5.3 Operating Characteristics of the Motor in Liquefied Natural Gas (–161°C)

As mentioned earlier, considering the safety of the test, the prototype is tested in liquid nitrogen, and it actually runs in LNG. Because the temperature and physical properties of liquid nitrogen and LNG are different, the parameters and rotor friction



loss of the prototype in the two liquids are also different (Ai et al., 2015). Using the aforementioned simulation model and the prototype parameters shown in **Table 2**, the operating performance of the prototype in LNG $(-161^{\circ}C)$ is analyzed, and the relevant curves are shown in **Figure 14**.

Compared with the calculation results of that running in LNG and liquid nitrogen, the operating characteristics of the prototype change little. In addition, its operating efficiency in LNG is slightly higher than that in liquid nitrogen.

6 CONCLUSION

An 18.5 kW CPMSM for a submerged LNG pump is studied. A CT design method is used to develop the CPMSM, the simulation modeling and analysis are carried out, and the liquid nitrogen test platform is built. The test results are very close to the simulation results.

Based on the theory of electric machinery and using the simulation and experimental data, this study also makes further analysis and research on the operational performance of the CPMSM and draws the following conclusion:

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- (1) The CPMSM can maintain high efficiency and high power factor operation within a wide range of output power.
- (2) The torque coefficient *K*t of the CPMSM is slightly higher than that at normal temperature when the magnetic circuit is not included.
- (3) Although the current tested in liquid nitrogen is about 1.6 times that in LNG due to the physical difference between the two liquids, there is little difference in the operating characteristics of the CPMSM in liquid nitrogen and LNG. Therefore, it is reasonable to use liquid nitrogen for testing the CPMSM.

With the continuous development and improvement of the magnetic material, design, and control technology of CPMSM, it will be better applied in the submerged LNG pumps.

DATA AVAILABILITY STATEMENT

The raw data supporting the conclusion of this article will be made available by the authors, without undue reservation.

AUTHOR CONTRIBUTIONS

Conceptualization, CG and YF; methodology, SH; software, CG; validation, JW and YF; formal analysis, JW; investigation, CG; resources, JW; data curation, CG; writing—original draft preparation, CG; writing—review and editing, YF; visualization, JW and YF; supervision, SH; project administration, SH; funding acquisition, YF. All authors have read and agreed to the published version of the manuscript.

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