



Cross Laminated Timber (CLT) Beams Loaded in Plane: Testing Stiffness and Shear Strength

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Cross Laminated Timber (CLT) is a relatively new timber product used in construction that has gained popularity over the last decade. The product itself is constituted by multiple glued layers of juxtaposed boards, usually arranged in an orthogonal direction between one layer and the adjacent ones. This particular structure brings several benefits, such as the possibility to use the same product both for walls and slabs, since it can bear in-plane and out-of-plane loads. However, the mechanical behavior differs from usual timber products, and research is still ongoing to achieve common agreement on standard procedures for testing products and theories for evaluating stresses for safety verifications. This paper focuses on the in-plane shear behavior of CLT and analyzes the existing methods to evaluate shear stresses. An experimental part then presents a four-point bending test of CLT beams with a specific geometry to induce shear failure. Results are reported both for the elastic range test, measuring the Modulus of Elasticity, and for the failure test to investigate shear behavior with regard to different mechanisms. Previously exposed methods are used for the calculation of shear stresses and to analyze the correspondence between them, and the results are then compared with other existing tests and values in literature. A new test setup for future research is eventually proposed.

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1. INTRODUCTION

Cross Laminated Timber is one of the many systems of building with timber, and it can be seen as a development of glued laminated timber by applying a similar concept on 2D elements instead of linear elements and with a new layout; its employment in construction is recent and it has become widely used mostly in Europe over the past 15 years. This product is usually produced in a plate-like shape and its alternated orthogonal board layer structure makes it apt to bear loads in and out of plane; hence the great benefit given by the possibility to use the same element both as a wall and as a floor slab. Another big advantage of this building system is the high degree of prefabrication it offers; this means more control during the production process, which translates to small tolerances. CLT is produced and, if necessary, cut in personalized shapes directly at the production site using CNC machines. Regarding the building site, this also means faster times and cleaner area, since the elements only need to be assembled and connected to each other to constitute the load-bearing structure of the building; it further permits faster application of additional insulation layers and finishes (see Brandner et al., 2016). The use of this construction system is relatively new, so the

TABLE 1 | Characteristics of tested CLT beams.

Series	n. of spec.	$b_{l,mean}$ [mm]	t_{CL} [mm]	t_i [mm]	Edge gluing
A3	4	100	90	30-30-30	Yes
A5	2	100	130	29-21-29-21-29	Yes
B5	2	80	135	27-27-27-27-27	No
C5	2	150	144	34-21-34-21-34	No

process of producing standards is still ongoing. The problem regards two different but closely related areas: which test procedures to use for the evaluation of the strength properties of CLT and also which methods to adopt for the calculation of stresses. The work of this paper is placed within this scope, particularly regarding the in-plane shear properties; a four-point bending test has been performed on CLT beams according to the EN 408 (2012) procedure to investigate the values of the modulus of elasticity and shear stresses at failure. On the matter of evaluating shear stresses, a review is provided presenting the available methods in literature, and then a test is also used for comparison.

2. MATERIALS

The experimental campaign investigated a total number of 10 CLT beams, coming from different producers. The beams were cut from bigger panels and geometry was modified from the prescriptions of EN 408 (2012) to induce a shear failure. All beams had a span $l = 3$ m and a height $h_{CL} = 600$ mm, the thickness varied depending on the number of layers and the producer, as seen in **Table 1**. Four different types of specimen have been tested, with differences in the number and thickness of layers, presence of narrow edge glued interface, board width and presence of cracks or cuts to improve shrinkage behavior, see **Figure 1**. For all panels the technical certificate of the producer indicated a minimum of 90% C24 strength class boards for each layer, with a maximum 10% of C16 boards (for strength classes of structural timber in Europe see EN 338, 2016).

3. TEST SETUP

The beams were tested using a four point bending test in accordance to the procedures of EN 408 (2012) and EN 16351 (2015), see **Figure 2**. The test was conducted in a displacement control method with a loading ratio of 0,03 mm/s until reaching a maximum force $F \leq 0,4 F_{max,est}$ to obtain the load/displacement curve in order to calculate the modulus of elasticity. The load was applied by a hydraulic actuator, and the integrated load cell was used to measure the force. A rigid steel beam was used to distribute equally the load in two centered points at a distance $c = 1100$ mm ($a = 700$ mm; $c = 800$ mm and $a = 850$ mm only for the first two specimens of series A3); in order to avoid local concentration of stresses, the load was transferred with two steel plates screwed on the beam. The beam itself was simply supported by two concrete foundations,

with one end functioning as a slider allowing movement in the longitudinal axis direction. Given the particular slenderness of the beam, two additional restraints were placed to avoid instability out of plane, with internal surfaces of frictionless plastic material in order to allow free movement of the beam in its bending plane. Displacement measures were taken with 4 LVDT on each side of the beam; three of them measured absolute displacements at neutral axis height (one in the center and two additional centered with $l_1 = 800$ mm, $l_1 = 600$ mm only for the first two specimen of series A3) while the fourth measured relative displacement.

4. MODULUS OF ELASTICITY CALCULATION

The modulus of elasticity was calculated in accordance with EN 408 (2012). Experimental load-displacement curves were analyzed through a linear regression analysis, and for each specimen the modulus was calculated considering the longest line between $0,1 F_{max}$ and $0,4 F_{max}$ with a minimum correlation coefficient of 0,99 (the line must at least include the interval between $0,2 F_{max}$ and $0,3 F_{max}$). The equation used is the following:

$$E_{m,l} = \frac{al_1^2(F_2 - F_1)}{16I_{net}(w_2 - w_1)} \quad (1)$$

- $(F_2 - F_1)$: load increase [N]
- $(w_2 - w_1)$: displacement increase in the corresponding interval [mm]
- a : distance between the loading point and the support [mm]
- l_1 : reference length for MoE determination [mm]
- I_{net} : moment of inertia referred to net section of the beam (layers parallel to x axis) [mm⁴].

5. SHEAR STRESSES CALCULATION

In this section various methods for calculating in-plane shear stresses will be presented and compared: a method based on the equilibrium, developed by Andreolli et al. (2012) (Equilibrium method), a method based on the Representative Sub Element Volume method, developed by Bogensperger et al. (2010) (RVSE method), the method for beams developed by Flaig and Blass (2013) and adopted also in the COST document (Brandner et al., 2018, Beam method), and the method of Austrian annex K to ÖNORM B 1995-1-1 (2015) (AT-annex method). In order to obtain a simple and significant exposition, a common notation will be used—the one present in the COST document (Brandner et al., 2018; Dietsch et al., 2018), see **Figure 3**—which will also be the base for the future Eurocode section regarding CLT products. The first index indicates the plane normal to the action, the second index indicates the axis the action is parallel to; for the following section then the direction y would be the vertical one (parallel to the height h_{CL} of the beam), x is horizontal (parallel to the longitudinal axis of the beam), z is horizontal through the

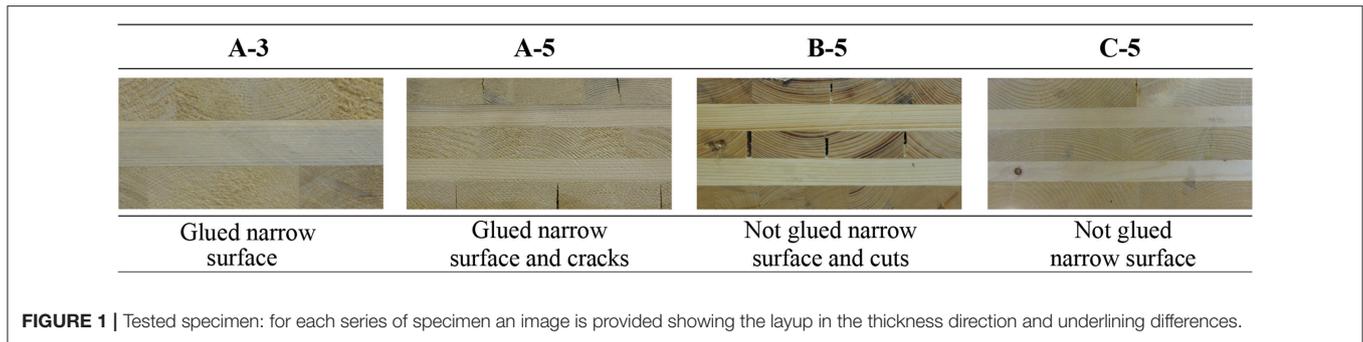


FIGURE 1 | Tested specimen: for each series of specimen an image is provided showing the layup in the thickness direction and underlining differences.

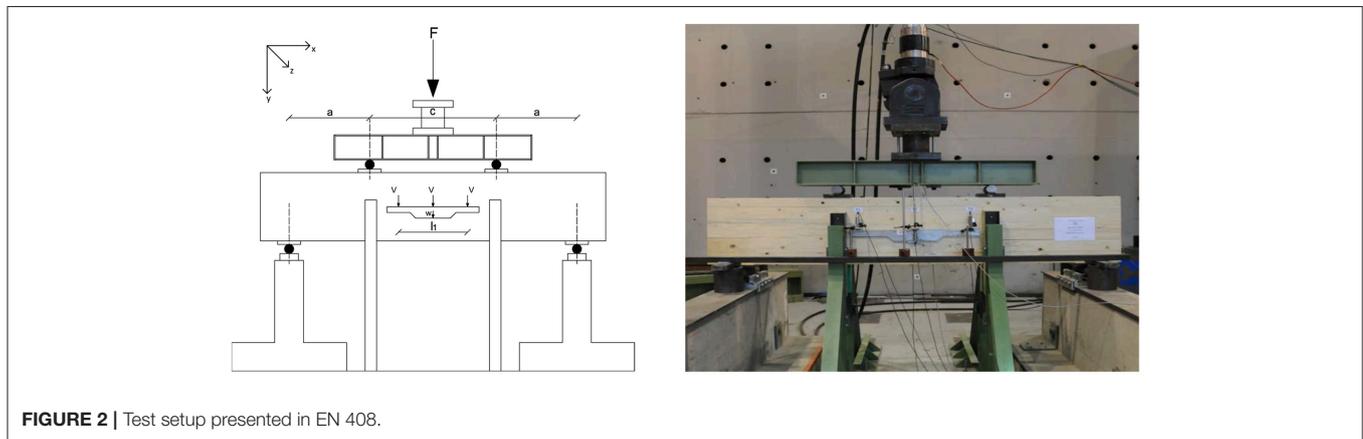


FIGURE 2 | Test setup presented in EN 408.

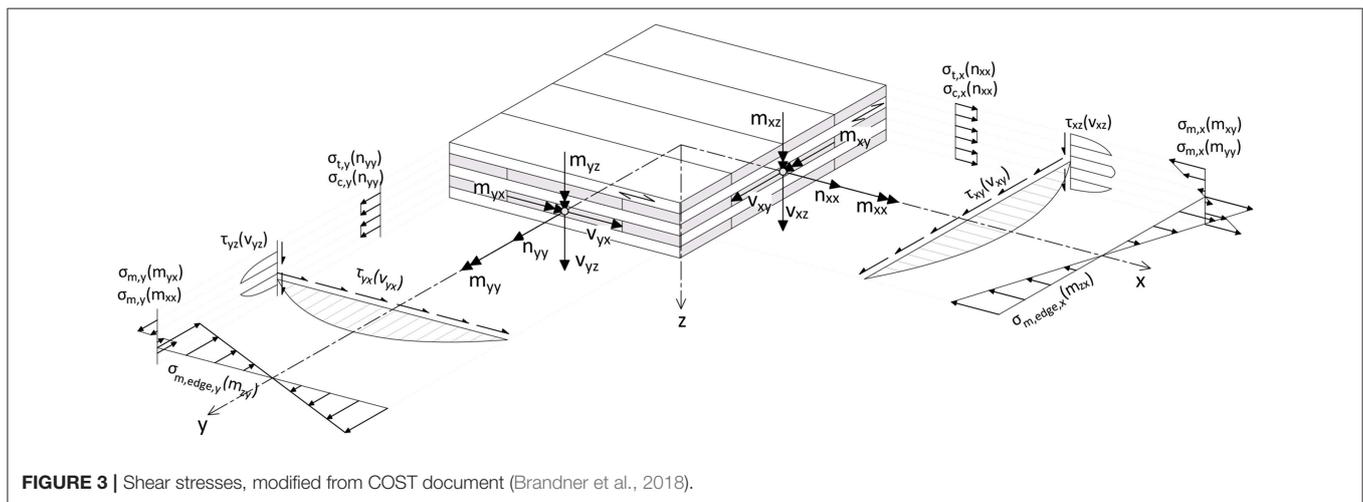


FIGURE 3 | Shear stresses, modified from COST document (Brandner et al., 2018).

thickness t_{CL} of the beam. The shear force is expressed as force per unit length and for rotational equilibrium:

$$v_{xy} = v_{yx} = v \tag{2}$$

The layers oriented as the x axis (so the major number of layers for a usual panel with an odd total number of layers) will have thickness t_1 t_3 t_5 , while the layers oriented as the y axis (so the minor number of layers) will have thickness t_2 t_4 . The width of

the laminations b_l is assumed to be equal for boards oriented in both directions, if cracks are present or it is not constant for each board then $b_{l,mean}$ is to be used (unless otherwise specified).

5.1. Equilibrium Method

This method is based on equilibrium equations for each layer and glued interfaces and was presented in Andreolli et al. (2012). As seen from **Figure 4** the base assumption is that shear stresses

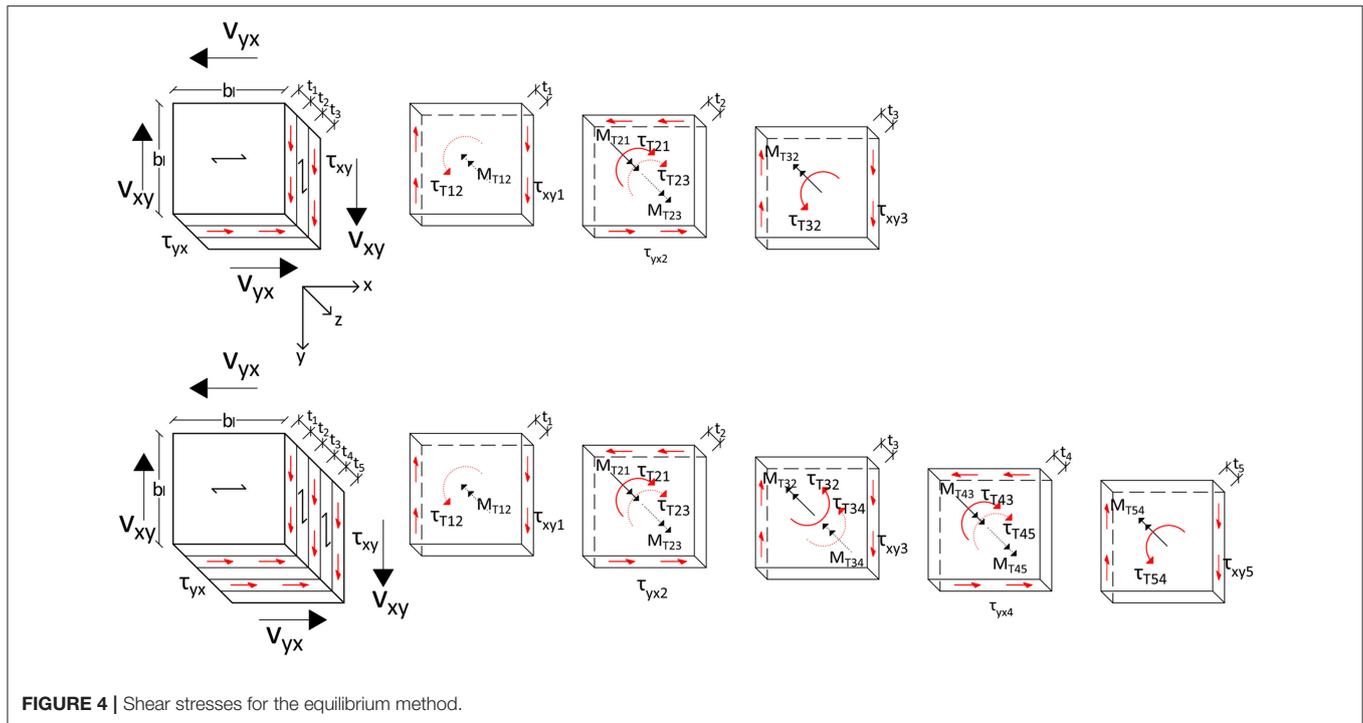


FIGURE 4 | Shear stresses for the equilibrium method.

are only present in the cross section of the boards oriented perpendicular in respect to the shear action.

5.1.1. 3 Layer Panel

Shear stresses τ_{xy} and τ_{yx} are calculated using the thicknesses of the layers oriented as the respective direction:

$$\tau_{xy} = \frac{v}{t_1 + t_3} \tag{3}$$

$$\tau_{yx} = \frac{v}{t_2} \tag{4}$$

For panels with symmetric layup $t_1 = t_3$ the stress τ_{yx} can be expressed as a function of τ_{xy} :

$$\tau_{yx} = \tau_{xy} \cdot \frac{2 \cdot t_1}{t_2} \tag{5}$$

The global equilibrium to rotation poses:

$$M_{T12} - M_{T21} - M_{T23} + M_{T32} = 0 \tag{6}$$

At each glued interface, for action reaction:

$$\begin{cases} M_{T12} = M_{T21} \\ M_{T23} = M_{T32} \end{cases} \tag{7}$$

Rotational equilibrium is calculated for each layer:

$$\begin{cases} M_{T12} - \tau_{xy1} \cdot b_l^2 \cdot t_1 = 0 \\ M_{T21} - \tau_{yx2} \cdot b_l^2 \cdot t_2 + M_{T23} = 0 \\ M_{T32} - \tau_{xy3} \cdot b_l^2 \cdot t_3 = 0 \end{cases} \tag{8}$$

So, considering that $\tau_{xy1} = \tau_{xy3} = \tau_{xy}$ and $\tau_{yx2} = \tau_{yx}$, and using the relations previously found, the following can be obtained:

$$M_T = M_{T12} = M_{T21} = M_{T23} = M_{T32} = \tau_{xy} \cdot b_l^2 \cdot t_1 \tag{9}$$

It is then possible to evaluate torsional shear stresses as a function of τ_{xy} considering that $W = W_1 = W_2 = W_3 = \frac{b_l^3}{3}$:

$$\tau_T = \tau_{T12} = \tau_{T21} = \tau_{T23} = \tau_{T32} = \frac{M_T}{W} = 3 \cdot \frac{\tau_{xy} \cdot t_1}{b_l} \tag{10}$$

5.1.2. 5 Layer Panel

Shear stresses τ_{xy} and τ_{yx} are calculated using the thicknesses of the layers oriented as the respective direction:

$$\tau_{xy} = \frac{v}{t_1 + t_3 + t_5} \tag{11}$$

$$\tau_{yx} = \frac{v}{t_2 + t_4} \tag{12}$$

For panels with symmetric layup $t_1 = t_3 = t_5$ and $t_2 = t_4$ the stress τ_{yx} can be expressed as a function of τ_{xy} :

$$\tau_{yx} = \tau_{xy} \cdot \frac{3t_1}{2t_2} \tag{13}$$

Similarly to the previous 3 layer case the same calculations are done, arriving at:

$$\begin{cases} M_{T,ext} = M_{T12} = M_{T21} = M_{T45} = M_{T54} = \tau_{xy} \cdot b_l^2 \cdot t_1 \\ M_{T,int} = M_{T23} = M_{T32} = M_{T34} = M_{T43} = \frac{\tau_{xy}}{2} \cdot b_l^2 \cdot t_1 \end{cases} \tag{14}$$

It is then possible to evaluate torsional shear stresses as a function of τ_{xy} considering that $W = W_1 = W_2 = W_3 = W_4 = W_5 = \frac{b_l^3}{3}$:

$$\begin{cases} \tau_{T,ext} = \tau_{T12} = \tau_{T21} = \tau_{T45} = \tau_{T54} = \frac{M_{T,ext}}{W} = 3 \cdot \frac{\tau_{xy} \cdot t_1}{b_l} \\ \tau_{T,int} = \tau_{T23} = \tau_{T32} = \tau_{T34} = \tau_{T43} = \frac{M_{T,int}}{W} = \frac{3}{2} \cdot \frac{\tau_{xy} \cdot t_1}{b_l} \end{cases} \quad (15)$$

So, differently from the case of a 3 layer panel torsional shear stresses are not equal for all glued interfaces but are major on external ones.

5.2. RVSE Method

This model is developed by referring to an ideal CLT panel with an infinite number of layers and considering a crossing interface with width equal to the width of the laminations, see Bogensperger et al. (2010). This element is then simplified to obtain a Representative Volume Sub Element (RVSE) of CLT, which has a thickness t and a nominal shear stress τ_0 distributed on its entire thickness t (see Figure 5):

$$\tau_0 = \frac{v}{t} \quad (16)$$

This stress can be considered as the composition of two parts: an effective shear stress on the cross section with orientation perpendicular to grain

$$\tau_v = 2 \cdot \tau_0 \quad (17)$$

and a torsional shear stress

$$\tau_T = 3 \cdot \tau_0 \cdot \frac{t}{b_l} \quad (18)$$

Since the real CLT panel has a finite number of layers it is necessary to refer to fictitious thicknesses of RVSE t_i^* ($n_{CA} = n_{lay} - 1$ is the number of glued interfaces):

$$t_{tot}^* = \sum_1^{n_{CA}} t_i^* \quad (19)$$

$$t_i^* = \begin{cases} \min(2 \cdot t_1; t_2) & \text{for the case of } t_1 \text{ external layer and } t_2 \text{ internal} \\ \min(t_3; t_4) & \text{for the case of } t_3 \text{ and } t_4 \text{ both internal layers} \end{cases} \quad (20)$$

So for a real CLT panel we get:

$$\tau_0^* = \frac{v}{t_{tot}^*} \quad (21)$$

$$\tau_{yx} = \tau_v^* = 2 \cdot \tau_0^* \quad (22)$$

$$\tau_T = \tau_T^* = 3 \cdot \tau_0^* \cdot \frac{t_i^*}{b_l} \quad (23)$$

It can be seen that τ_v^* corresponds to the major of the shear stresses calculated before with the equilibrium method, which is τ_{yx} for “usual” cases of symmetrical CLT panels with an odd number of layers and where the total thickness t_x of layers oriented as x axis is larger or equal to the total thickness t_y of the other layers oriented as y axis. Under the same assumptions, it is also true that the torsional shear stress calculated with this method coincides precisely with the one calculated for a 3 layer panel with the equilibrium method, while for a 5 layer panel the result of the RVSE method is the average of the two values obtained with the equilibrium method. A detailed demonstration of these observations is provided in the Annex.

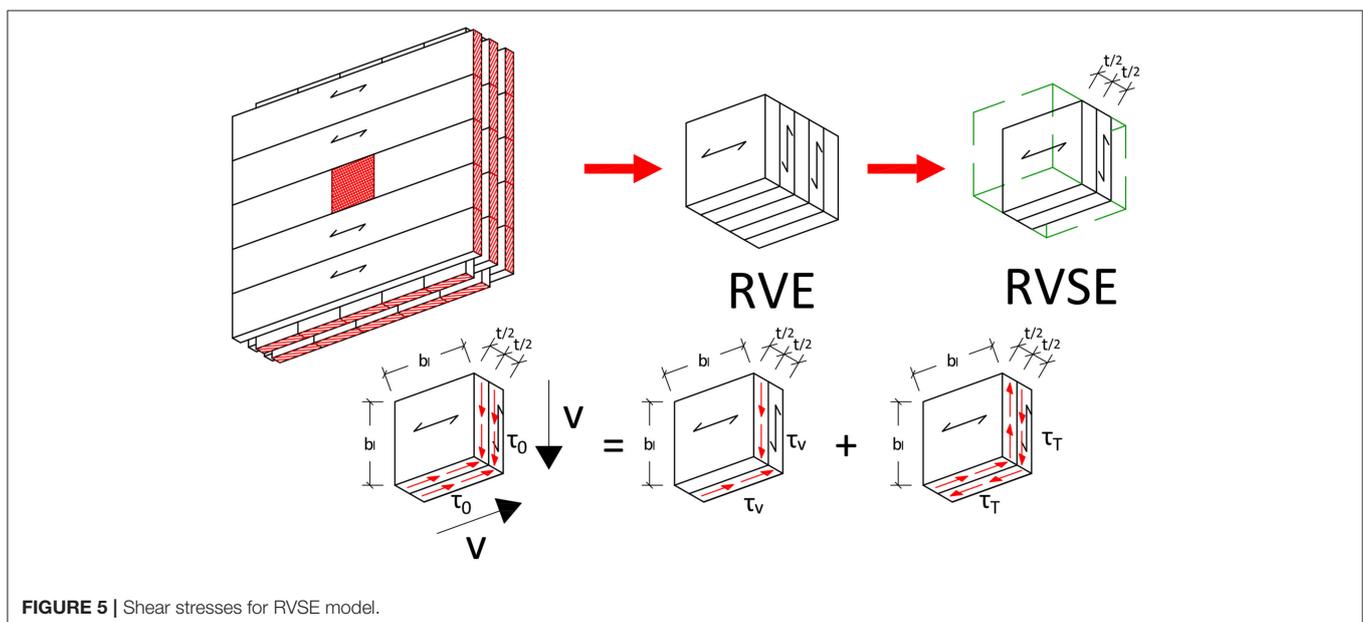


FIGURE 5 | Shear stresses for RVSE model.

5.3. Beam Method

This method was developed in Flaig and Blass (2013) by referring to the equilibrium of a beam and is also present in the COST document (Brandner et al., 2018; Dietsch et al., 2018) which will be the base for a new part in the EN 1995 regarding CLT products. For shear stresses the calculus is carried out separately considering net areas for both directions; the only difference between Flaig theory and COST document is that the latter advises reducing by 0,20 the area comprising outer layers. For a 3 layer panel:

$$\left\{ \begin{array}{l} \tau_{xy} = \frac{v}{t_1 + t_3} \quad \text{Flaig} \\ \tau_{xy} = \frac{v}{0,8 \cdot (t_1 + t_3)} \quad \text{COST} \end{array} \right. \quad \left\{ \begin{array}{l} \tau_{yx} = \frac{v}{t_2} \quad \text{Flaig} \\ \tau_{yx} = \frac{v}{t_2} \quad \text{COST} \end{array} \right. \quad (24)$$

And for a 5 layer panel:

$$\left\{ \begin{array}{l} \tau_{xy} = \frac{v}{t_1 + t_3 + t_5} \quad \text{Flaig} \\ \tau_{xy} = \frac{v}{0,8 \cdot (t_1 + t_5) + t_3} \quad \text{COST} \end{array} \right. \quad \left\{ \begin{array}{l} \tau_{yx} = \frac{v}{t_2 + t_4} \quad \text{Flaig} \\ \tau_{yx} = \frac{v}{t_2 + t_4} \quad \text{COST} \end{array} \right. \quad (25)$$

With this method then the shear stresses are exactly the same as the Equilibrium method (except for the 0,20 reduction factor in the COST document). Torsional shear stresses are calculated as:

$$\tau_T = \frac{3V_{xy}}{b_l^2 \cdot n_{CA}} \left(\frac{1}{n_l} - \frac{1}{n_l^3} \right) \quad (26)$$

where V_{xy} is the applied shear force, $n_{CA} = n_{lay} - 1$ is the number of glued interfaces, $n_l = \frac{h_{CL}}{b_l}$ is the number of laminations in the height of the beam. This formula gives torsional shear stress values very close to the ones of the RVSE method, and for high n_l values they coincide, for the limit case of indeed $1/n_l^3 \rightarrow 0$:

$$\begin{aligned} \tau_T &= \frac{3V_{xy}}{b_l^2 \cdot n_{CA}} \cdot \frac{1}{n_l} = \frac{3V_{xy}}{b_l^2 \cdot n_{CA}} \cdot \frac{1}{\frac{h_{CL}}{b_l}} = \frac{3V_{xy}}{b_l \cdot h_{CL} \cdot n_{CA}} \quad (27) \\ &= 3 \cdot \frac{v}{b_l \cdot n_{CA}} \end{aligned}$$

Remembering Equations (19) and (21) and with the consideration that $t_{tot}^*/n_{CA} = t_i^*$, it is possible to obtain the same formula of the RVSE method (Equation 23):

$$\tau_T = 3 \cdot \frac{v}{b_l \cdot n_{CA}} = 3 \cdot \frac{\tau_0^* \cdot t_{tot}^*}{b_l \cdot n_{CA}} = 3 \cdot \tau_0^* \cdot \frac{t_i^*}{b_l} \quad \square \quad (28)$$

Two additional shear stresses on the glued interface are presented in this method, as seen in **Figure 6**, which are not regarded in the other methods (so they will not be considered for the comparison with the other methods in the next section): τ_{zx} which lies in the intersection plane (z) and is parallel to the axis of the beam (x), and τ_{zy} , which lies in the same plane and is parallel to the axis of the beam height (y):

$$\tau_{zx} = \frac{6V_{xy}}{b_l^2 \cdot n_{CA}} \left(\frac{1}{n_l^2} - \frac{1}{n_l^3} \right) \quad (29)$$

$$\tau_{zy} = \frac{q}{n_l \cdot b_l} \quad (30)$$

5.4. AT-Annex

The Austrian Annex K to ÖNORM B 1995-1-1 (2015) proposes, regarding shear stresses, a verification with reference to the net area in the two directions:

$$\tau_{V,0,d} = \frac{n_{xy,d}}{\min(A_x; A_y)} \quad (31)$$

which then yields, with the previous conventions:

$$\tau_{xy} = \frac{v}{t_1 + t_3} \quad \tau_{yx} = \frac{v}{t_2} \quad \text{3 layer panel} \quad (32)$$

$$\tau_{xy} = \frac{v}{t_1 + t_3 + t_5} \quad \tau_{yx} = \frac{v}{t_2 + t_4} \quad \text{5 layer panel} \quad (33)$$

In this case then, these values are exactly the same as the previous methods. For what regards torsional shear stresses the proposed formulation is (adopting the previous conventions, where for usual panels the major shear stress is τ_{yx}):

$$\tau_T = 3 \cdot \tau_{yx} \cdot \frac{t_{i,max}}{b_l} \quad (34)$$

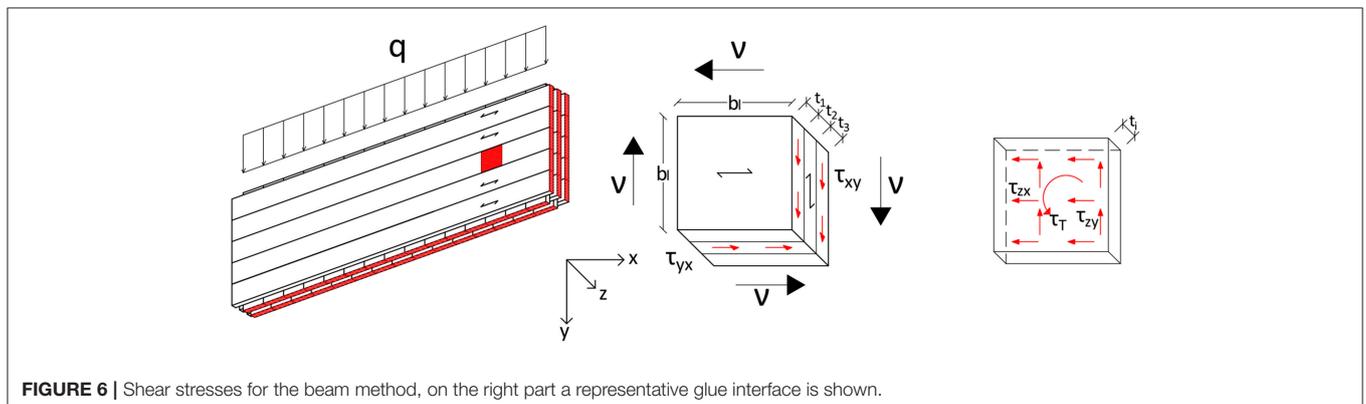


FIGURE 6 | Shear stresses for the beam method, on the right part a representative glue interface is shown.

TABLE 2 | Modulus of Elasticity values [GPa].

Series	MoE [GPa]				Mean
	Spec. 1	Spec. 2	Spec. 3	Spec. 4	
A3	–	–	12,298	10,997	11,648
A5	15,845	14,758	–	–	15,302
B5	14,445	16,661	–	–	15,553
C5	12,493	10,775	–	–	11,634

The formula can be developed for a comparison:

$$\tau_T = 3 \cdot \frac{\nu}{m_{lay,y} \cdot t_{l,y}} \cdot \frac{t_{i,max}}{b_l} \quad (35)$$

$$\tau_T = 3 \cdot \frac{\nu}{t_{l,y}} \cdot \frac{t_{i,max}}{b_l} \quad \text{3-layer CLT} \quad (36)$$

$$\tau_T = \frac{3}{2} \cdot \frac{\nu}{t_{l,y}} \cdot \frac{t_{i,max}}{b_l} \quad \text{5-layer CLT}$$

Equilibrium method

$$\tau_T = \frac{3}{2} \frac{\nu}{b_l} \quad \text{3-layer CLT} \quad \tau_{T,mean} = \frac{3}{4} \frac{\nu}{b_l} \quad \text{5-layer CLT} \quad (37)$$

The two formulations are similar, but the results obtained are quite different: one considers the maximum lamination thickness and the other the mean, and even when these two values coincide (i.e., when all layers have equal thickness) the Austrian formulation gives values which are exactly double those of the equilibrium method.

6. BENDING STRESS CALCULATION

Regarding bending stresses there is common agreement on the method of calculation, so with the usual convention of **Figure 3** the formula used for the maximum bending stress is:

$$\sigma_{m,edge,x} = \frac{M_{xz}}{I_{net}} \cdot \frac{h_{CL}}{2} \quad (38)$$

- I_{net} : moment of inertia referred to net section of the beam (layers parallel to x axis) [mm⁴]
- h_{CL} : height of the CLT beam [mm].

7. EXPERIMENTAL RESULTS

In **Table 2** the results of the elastic part of the test are presented in terms of Modulus of Elasticity, calculated as explained in section 4. For the Series A3 the first specimen was only tested to failure and the second one was discarded since it did not respect all prescriptions of EN 408 (2012).

The results regarding failure are instead presented in **Table 3** in terms of the maximum force applied by the press, so then $V_{max} = V_{xy}$ and $M_{max} = M_{xz}$.

TABLE 3 | Values of applied force and relative shear and bending moment at failure.

Series	Fmax [kN]				Mean	Vmax [kN]	Mmax [kNm]
	Spec. 1	Spec. 2	Spec. 3	Spec. 4			
A3	313	324	372	310	330	165	128
A5	506	515	–	–	511	255	179
B5	417	405	–	–	411	206	144
C5	565	495	–	–	530	265	186

TABLE 4 | Stresses at failure, shear stresses calculated with the equilibrium method.

Series	Stresses at failure [MPa]					Failure
	$\sigma_{m,edge,x}$	τ_{xy}	τ_{yx}	$\tau_{T,ext}$	$\tau_{T,int}$	
A3	35,42	6,88	13,75	6,19	/	Torsional
A5	34,20	7,34	15,21	6,39	3,19	Torsional
B5	29,59	6,34	9,51	6,42	3,21	Torsional
C5	30,31	6,50	15,77	4,42	2,21	Bending

The result in terms of bending and shear stresses are presented in **Table 4**, using the mean values of moment and shear force at failure. For the calculation of shear stresses the equilibrium method was used, and to obtain the maximum value a Jourawski distribution was assumed.

It is also interesting to see from **Figure 7** a comparison between the values of shear stresses evaluated using all the different methods previously explained. Shear stress τ_{xy} is the same value for all methods, except for the COST method, which results in a slightly higher value due to a 0,20 reduction factor for the thickness of outer layers. Shear stress τ_{yx} is the same value for all methods, while torsional shear stresses present some differences. The equilibrium method presents two different values, one for external and one for internal interfaces while the other methods present a single value which for the RVSE, COST and Beam methods is more or less the average of the previous values, while for the Austrian Annex the value is much higher.

8. DISCUSSION AND COMPARISON

The different failures for the four types of specimen can be seen in **Figure 8**. No specimen failed due to shear stresses in the laminations, but two other types of failures were observed: specimens A3, A5, and B5 all failed due to torsional shear stresses in the glued interfaces, while specimen C5 failed in bending. From the figures, another interesting detail appears: torsional shear failure in 5 layer panels (A5 and B5) started from the outer glued interfaces, which is in accordance with the equilibrium method that hypothesizes different values of torsional shear stresses, major in external glued interfaces and minor in internal ones. Specimen series C5 failed in bending, and this is probably due to a stress concentration near a defect present in both beams which brought to an early bending failure. The

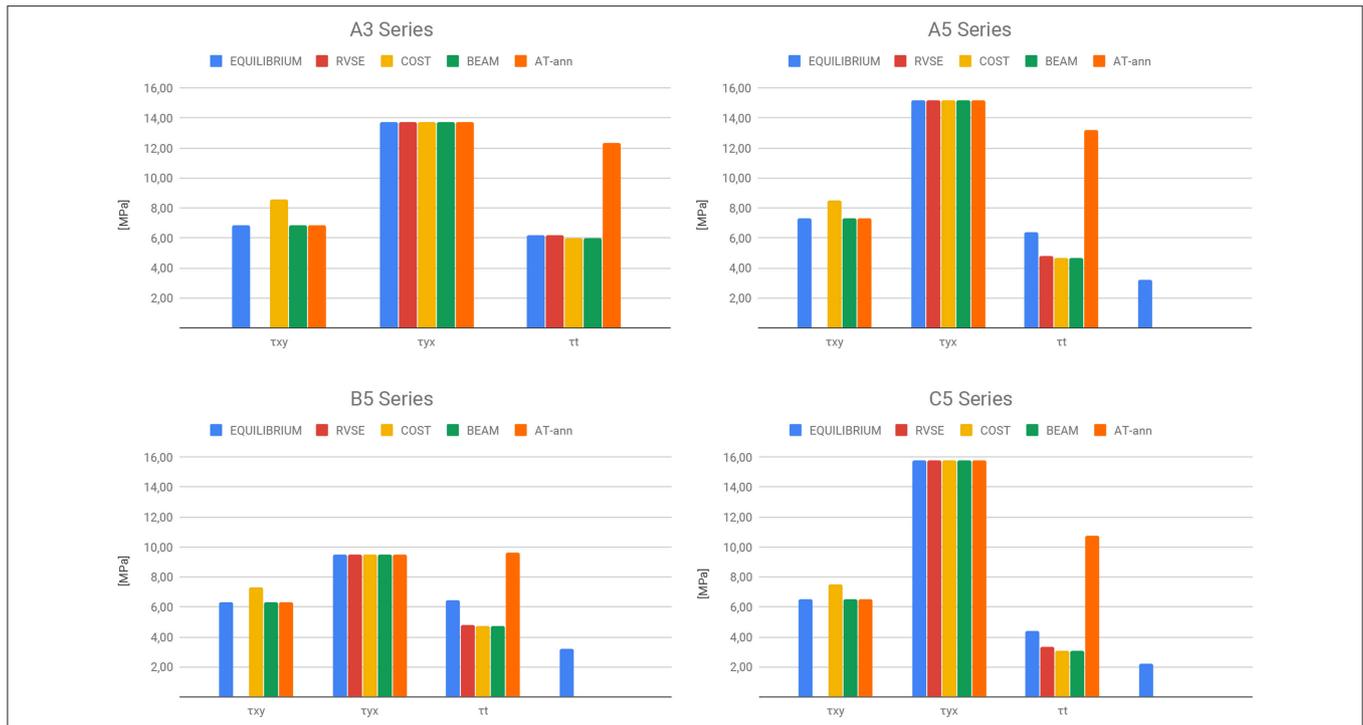


FIGURE 7 | Comparison of shear stresses at failure. Note that for the RVSE method there is no value for τ_{xy} since this method only provides the value of major stress τ_{yx} . Note also that for 5 layer panels the equilibrium method provides two values of τ_t , one for the external and one for the internal glued interface.

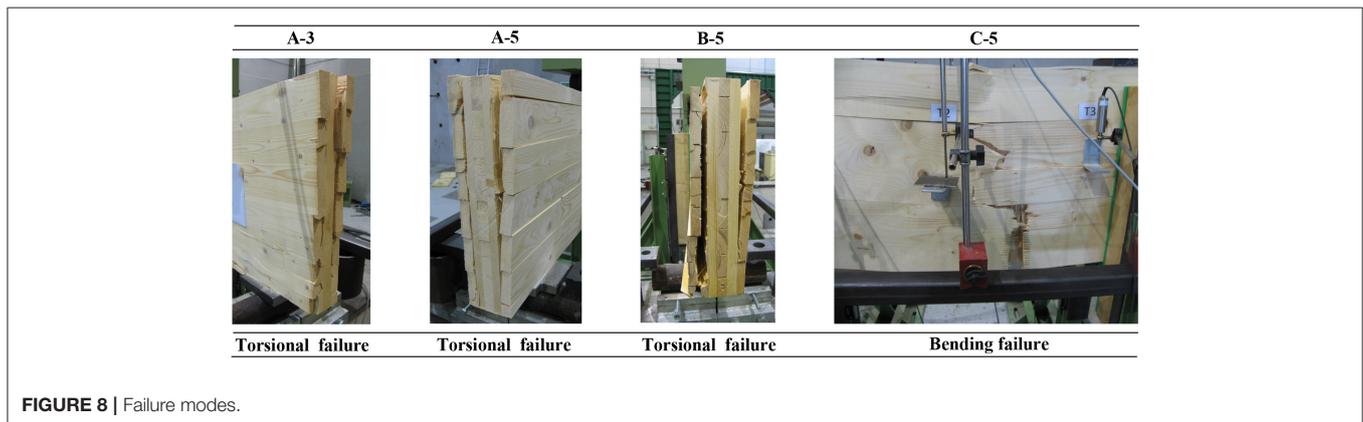


FIGURE 8 | Failure modes.

C5 series was also the one with the largest width of laminations, thus it had the highest polar moment of inertia which also contributed to prevent torsional shear failure like in the other three cases. The presence or absence of glue on the narrow surface of laminations does not seem to have an influence on the values of torsional shear stresses of failure, as seen from the values in **Table 4**.

A comparison can be done with stress values from other authors present in the literature. Regarding bending stresses at failure a value $f_{m,mean} = 38,5$ MPa was obtained in Jöbstl et al. (2008) using the same four-point bending test procedure used in this article; the authors obtained bending failure for all specimens so the value obtained seems reliable and is indeed higher than all

bending stresses in **Table 4**. In the same paper a different setup is also tested (from Wallner, 2004), presenting two possible failure sections which proved to be effective in obtaining shear failure in the lamellas. The value obtained for shear stresses at failure was $f_{v,mean} = 12,8$ MPa, which is lower (except for B5 series) than the values obtained here. Two considerations can be done: the first is that, in the present testing campaign, shear failure in the lamellas was not obtained. The second is that the test devised in Wallner (2004) presents a very specific geometry whose capacity to represent real cases of CLT beams and walls is difficult to prove. Another different test setup, based on a particular configuration with orientation of 14° in respect to the compression force, is presented in Brandner et al. (2013) (from Hirschmann, 2011)

which achieves shear failure in lamellas with a value $f_{v,\text{mean}} = 9$ MPa. Another test which succeeded in obtaining shear failure in lamellas is the one presented in Brandner et al. (2017) (taken also from Kreuzinger and Sieder, 2013; Dröscher, 2014; Brandner et al., 2015). It contemplates a compression test on a column specimen which has grain orientation at an angle in respect to the applied force; results are provided for a series tested at TU Graz and another at TU Munich, which are, respectively, $f_{v,\text{mean}} = 7,3$ MPa and $f_{v,\text{mean}} = 7,6$ MPa. Regarding torsional stresses, the majority of tests found in the literature are on a single crossing interface, such as the ones present in Blaß and Görlacher (2001) and Jöbstl et al. (2004) (from Jeitler, 2004). Torsional stress values at failure obtained from these two works are, respectively, $f_{t,\text{mean}} = 3,6$ MPa and $f_{t,\text{mean}} = 3,5$ MPa, which are well below the values obtained in the present article. This suggests probably that torsional shear strength is much higher in real scale CLT panels, which implies that torsional tests on single nodes may not be representative of the complexity of a complete CLT panel. It is also worth noting that in this paper a Jourawski shear stress distribution was assumed, thus the 1,5 factor may imply an overestimation of real stress distribution, both for shear stresses in the lamellas and for torsional shear stresses at the glue interface.

9. CONCLUSION AND FUTURE WORKS

In-plane shear stresses for CLT remain an open topic regarding which method to use for their evaluation and the test setup to measure strength values. This is due to the particular structure of this timber product which, differently from other simpler products like solid wood and GLT, presents different types of failure depending on loading, geometry and layout. In this paper a review and comparison between the available methods to calculate in-plane shear stresses for CLT panels was presented, with particular effort directed at trying to make uniform the notation for all methods to match the one of **Figure 3**, which will be probably used for the new Eurocode. From the comparison it is evident that for shear stresses in the laminations all methods provide the same values for usual CLT layouts, while regarding torsional shear stresses there are still some differences. The RVSE, COST and Beam methods all provide the same single

value for torsional shear stress, while the Equilibrium method provides two different values referred to external and internal glued interfaces (for 5 layers panels); for usual CLT layouts the average of these two values corresponds to the values of the previous three methods. The method proposed in the Austrian Annex to Eurocode instead provides much higher values than all other methods. An important topic for future developments could be to adopt a single method both for CLT walls and beams in the upcoming Eurocode, since for now two different methods are present in the draft, referring to the wall case or beam case (see Dietsch et al., 2018).

A four-point bending test was then applied to four different types of CLT beams to investigate in-plane shear behavior; in spite of the specifically chosen geometry no shear failure in the laminations was obtained—only torsional shear failure and bending failure in one case, which highlighted the inapplicability of such a testing setup to obtain information about shear strength. It is then necessary to devise a specific test capable of singling out the shear failure in laminations, and promising first results are coming from a test setup based on the diagonal compression of a CLT panel which will be presented in a future paper. Nevertheless, the high values of torsional shear stresses obtained at failure in this paper indicate much higher strength than the values present nowadays in the literature, underlying the importance of testing full-scale CLT panels and not simply conducting torsional tests on single nodes or crossing interfaces, even though this is presently still suggested in EN 16351 (2015).

DATA AVAILABILITY

The datasets generated for this study are available on request to the corresponding author.

AUTHOR CONTRIBUTIONS

MA, RT, and FB contributed conception and design of the study. MA performed the testing campaign. FB wrote the first draft of the manuscript. RT, MA, and FB wrote sections of the manuscript. All authors contributed to manuscript revision, read and approved the submitted version.

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Conflict of Interest Statement: The authors declare that the research was conducted in the absence of any commercial or financial relationships that could be construed as a potential conflict of interest.

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ANNEX

Comparison Between Equilibrium and RVSE Method

For a 3 layer panel n_{lay} denotes the total number of layers, $n_{lay,x}$ is the number of layers with grain oriented in the x direction, $t_{l,x}$ is the thickness of a single lamination oriented in the x direction):

$$n_{lay} = 3 \tag{39}$$

$$n_{lay,y} = \frac{n_{lay} - 1}{2} = 1 \tag{40}$$

$$n_{lay,x} = n_{lay} - n_{lay,y} = 2 \tag{41}$$

$$t_x = t_1 + t_3 = n_{lay,x} \cdot t_{l,x} = 2 \cdot t_{l,x}$$

HYP: symmetry $t_1 = t_3 = t_{l,x}$ (42)

$$t_y = t_2 = n_{lay,y} \cdot t_{l,y} = 1 \cdot t_{l,y} \quad t_2 = t_{l,y} \tag{43}$$

Equilibrium method

$$\tau_{xy} = \frac{v}{t_x} = \frac{v}{n_{lay,x} \cdot t_{l,x}} = \frac{v}{2 \cdot t_{l,x}} \tag{44}$$

$$\tau_{yx} = \frac{v}{t_y} = \frac{v}{n_{lay,y} \cdot t_{l,y}} = \frac{v}{t_{l,y}} \tag{45}$$

RVSE method

$$t_{tot}^* = \sum_{i=1}^{n_{lay}-1} t_i^* = \sum_1^2 t_i^* = t_{l,y} + t_{l,y} = 2 \cdot t_{l,y} \tag{46}$$

with this additional HYP: $t_x \geq t_y \rightarrow n_{lay,x} \cdot t_{l,x} \geq n_{lay,y} \cdot t_{l,y} \rightarrow t_{l,y} \leq 2 \cdot t_{l,x}$ (47)

the fictitious thickness is obtained as: $t_i^* = t_1^*$
 $= t_2^* = \min(2 \cdot t_{l,x}; t_{l,y}) = t_{l,y}$ (48)

$$\tau_0^* = \frac{v}{t_{tot}^*} = \frac{v}{2 \cdot t_{l,y}} \tag{49}$$

$$\tau_v^* = 2 \cdot \tau_0^* = 2 \cdot \frac{v}{2 \cdot t_{l,y}} = \frac{v}{t_{l,y}} \equiv \tau_{yx} \quad \square \tag{50}$$

For a 5 layer panels:

$$n_{lay} = 5 \tag{51}$$

$$n_{lay,y} = \frac{n_{lay} - 1}{2} = 2 \tag{52}$$

$$n_{lay,x} = n_{lay} - n_{lay,y} = 3 \tag{53}$$

$$t_x = t_1 + t_3 + t_5 = n_{lay,x} \cdot t_{l,x} = 3 \cdot t_{l,x}$$

HYP: symmetry and $t_3 = t_1$; so $t_1 = t_3 = t_5 = t_{l,x}$ (54)

$$t_y = t_2 + t_4 = n_{lay,y} \cdot t_{l,y} = 2 \cdot t_{l,y} \tag{55}$$

Equilibrium method

$$\tau_{xy} = \frac{v}{t_x} = \frac{v}{n_{lay,x} \cdot t_{l,x}} = \frac{v}{3 \cdot t_{l,x}} \tag{56}$$

$$\tau_{yx} = \frac{v}{t_y} = \frac{v}{n_{lay,y} \cdot t_{l,y}} = \frac{v}{2 \cdot t_{l,y}} \tag{57}$$

RVSE method

$$t_{tot}^* = \sum_{i=1}^{n_{lay}-1} t_i^* = \sum_1^4 t_i^* = t_{l,y} + t_{l,y} + t_{l,y} + t_{l,y} = 4 \cdot t_{l,y} \tag{58}$$

with this additional HYP: $t_{l,y} \leq t_{l,x}$ (59)

we obtain: $t_i^* = t_1^* = t_4^* = \min(2 \cdot t_{l,x}; t_{l,y}) = t_{l,y}$
 $t_i^* = t_2^* = t_3^* = \min(t_{l,x}; t_{l,y}) = t_{l,y}$ (60)

$$\tau_0^* = \frac{v}{t_{tot}^*} = \frac{v}{4 \cdot t_{l,y}} \tag{61}$$

$$\tau_v^* = 2 \cdot \tau_0^* = 2 \cdot \frac{v}{4 \cdot t_{l,y}} = \frac{v}{2 \cdot t_{l,y}} \equiv \tau_{yx} \quad \square \tag{62}$$

So for CLT panels (with an odd number of layers) where the layers oriented in the x direction have all the same thickness and this value is greater or equal than the thickness of the layers in the other direction there is coincidence between the two methods; these geometrical characteristics cover the majority of commercial CLT panels.

Keeping the same hypothesis made regarding shear stresses it is possible to compare torsional shear stresses. It can be seen that for a 3 layer panels it is the same value for both methods, while for a 5 layer panel a constant value is obtained from the RVSE method which is the average of the previous internal and external values of equilibrium method.

Equilibrium method

$$\tau_T = 3 \cdot \frac{\tau_{xy} \cdot t_1}{b_l} = 3 \cdot \frac{v}{n_{lay,x} \cdot t_{l,x}} \cdot \frac{t_{l,x}}{b_l} = \frac{3}{2} \frac{v}{b_l} \tag{63}$$

RVSE method

$$\tau_T^* = 3 \cdot \tau_0^* \cdot \frac{t_i^*}{b_l} = 3 \cdot \frac{v}{2 \cdot t_{l,y}} \cdot \frac{t_{l,y}}{b_l} = \frac{3}{2} \frac{v}{b_l} \equiv \tau_T \quad \square \tag{64}$$

For 5 layer panel Equilibrium method

$$\tau_{T,ext} = 3 \cdot \frac{\tau_{xy} \cdot t_1}{b_l} = 3 \cdot \frac{v}{n_{lay,x} \cdot t_{l,x}} \cdot \frac{t_{l,x}}{b_l} = \frac{v}{b_l} \tag{65}$$

$$\tau_{T,int} = \frac{3}{2} \cdot \frac{\tau_{xy} \cdot t_1}{b_l} = \frac{3}{2} \cdot \frac{v}{n_{lay,x} \cdot t_{l,x}} \cdot \frac{t_{l,x}}{b_l} = \frac{1}{2} \frac{v}{b_l} \tag{66}$$

$$\tau_{T,mean} = \frac{\tau_{T,ext} + \tau_{T,int}}{2} = \frac{3}{4} \frac{v}{b_l} \tag{67}$$

RVSE method

$$\tau_T^* = 3 \cdot \tau_0^* \cdot \frac{t_i^*}{b_l} = 3 \cdot \frac{v}{4 \cdot t_{l,y}} \cdot \frac{t_{l,y}}{b_l} = \frac{3}{4} \frac{v}{b_l} \equiv \tau_{T,mean} \quad \square \tag{68}$$

The raw data supporting the conclusions of this manuscript will be made available by the authors, without undue reservation, to any qualified researcher.

NOMENCLATURE

$\sigma_{m,edge,x}$	in plane bending stresses, referred to layers parallel to the grain of the outermost layers (x-direction)
$\sigma_{m,edge,y}$	in plane bending stresses, referred to layers perpendicular to the grain of the outermost layers (y-direction)
$\sigma_{c,x}$	compression stresses, referred to layers parallel to the grain of the outermost layers (x-direction)
$\sigma_{c,y}$	compression stresses, referred to layers perpendicular to the grain of the outermost layers (y-direction)
$\sigma_{m,x}$	out of plane bending stresses, referred to layers parallel to the grain of the outermost layers (x-direction)
$\sigma_{m,y}$	out of plane bending stresses, referred to layers perpendicular to the grain of the outermost layers (y-direction)
$\sigma_{t,x}$	tensile stresses, referred to layers parallel to the grain of the outermost layers (x-direction)
$\sigma_{t,y}$	tensile stresses, referred to layers perpendicular to the grain of the outermost layers (y-direction)
σ_{xy}	in plane shear stresses, referred to layers parallel to the grain of the outermost layers (x-direction)
σ_{xz}	out of plane shear stresses, referred to layers parallel to the grain of the outermost layers (x-direction)
τ_0	nominal shear stress (RVSE method)
τ_0^*	nominal shear stress, referred to real panel (RVSE method)
τ_v	effective shear stress (RVSE method)
τ_v^*	effective shear stress, referred to real panel (RVSE method)
$\tau_{T,ext}$	torsional stresses at glue interface, referred to external interfaces (Equilibrium method)
$\tau_{T,int}$	torsional stresses at glue interface, referred to internal interfaces (Equilibrium method)
τ_T	torsional stresses at glue interface
τ_{yx}	in plane shear stresses, referred to layers perpendicular to the grain of the outermost layers (y-direction)
τ_{yz}	out of plane shear stresses, referred to layers perpendicular to the grain of the outermost layers (y-direction)
b_l	width of laminations or mean distance between the edge and a groove or mean spacing between grooves within a lamination
$b_{l,mean}$	mean width of laminations or mean distance between the edge and a groove or mean spacing between grooves within a lamination
F	force
$F_{max,est}$	estimated maximum force at failure
F_{max}	maximum force at failure
h_{CL}	cross laminated timber height
I_{net}	moment of inertia referred to net section
l	length or span
M	bending moment
m	bending moment per unit length
M_T	torsional moment at glued interface
n	tension/compression force per unit length
n_l	number of laminations in the height of the beam
$n_{lay,x}$	number of layers in a cross laminated timber member with grain parallel to x-direction
$n_{lay,y}$	number of layers in a cross laminated timber member with grain parallel to y-direction
n_{lay}	number of layers of cross laminated timber member
n_{CA}	number of glued interfaces
$t_1 t_3 t_5$	thickness of each lamination parallel to the grain of outermost layers (x-direction)
$t_2 t_4$	thickness of each lamination perpendicular to the grain of outermost layers (y-direction)
t_i	thickness of a single lamination
t_i^*	fictitious thickness of a single lamination (RVSE method)
t_x	sum of thicknesses of layers in x-direction
t_y	sum of thicknesses of layers in y-direction
t_{CL}	cross laminated timber thickness
$t_{l,x}$	equal thickness of each layer in a cross laminated timber member with grain parallel to x-direction ($t_{l,x} = t_1 = t_3 = t_5$)
$t_{l,y}$	equal thickness of each layer in a cross laminated timber member with grain parallel to y-direction ($t_{l,y} = t_2 = t_4$)
V	shear force
v	shear force per unit length
W	torsional resistance moment