



An Efficient Track-Scale Model for Laser Powder Bed Fusion Additive Manufacturing: Part 2—Mechanical Model

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This is the second of two manuscripts that presents a computationally efficient full-field deterministic model for laser powder bed fusion (LPBF). The Hybrid Line (HL) thermal model developed in part I is extended to predict the in-process residual stresses due to laser processing of a nickel-based superalloy, RENÉ 65. The computational efficiency and accuracy of the HL thermo-mechanical model is first compared to the exponential decaying heat input model on a single-track simulation. LPBF thin-wall builds with three different laser powers and four printing patterns are evaluated in this study and compared with part-scale simulations. The simulations show good agreements with the experimental X-Ray diffraction measured residual stresses. Compared to the laser power, the scanning pattern is demonstrated to have significant effects on residual stresses. Laser scan patterns utilizing short laser paths generate lower tensile stress along the longitudinal direction of the part and higher compressive stress along the build direction.

Keywords: laser powder bed fusion, finite element modelling, residual stress, laser scanning pattern, superalloys

1 INTRODUCTION

Nickel-based (Ni-based) superalloys possess a combination of outstanding mechanical and physical properties at high temperatures, making them attractive for application in gas-turbine and jet-engine components (Reed 2006; Thatte et al., 2017; Stinville et al., 2018; Eftekhari et al., 2020). In aero-engine and gas turbine power industries, there are numerous geometrically complex components made with intricate serpentine cooling paths and thin wall sections, including the combustor, diffuser, and nozzle. Laser powder bed fusion (LPBF) is a promising route for the construction of near net shape, high-tolerance components capable of withstanding extreme environment and loading conditions (Carter et al., 2012). However, thin-wall Ni-based superalloys are susceptible to the formation of cracks and distortion during LPBF processing (Chakraborty et al., 2021).

The defects in superalloy LPBF parts are due to the rapid cooling rates reported to be on the order of 10^6 K/s during LPBF (Wang et al., 2019). These high cooling rates result in the formation of metastable microstructures with significant residual stresses (Lu et al., 2015; Li et al., 2018). Controlling and predicting the residual stresses during LPBF is not trivial. Both the magnitudes and distribution of residual stresses in AM components are governed by several factors, including: material properties, volumetric change due to phase transformation or precipitation, geometry of components, the position of specimens, processing parameters, baseplate temperature, and laser

scanning pattern (Bandyopadhyay and Traxel 2018; Withers and Bhadeshia 2001a; Withers and Bhadeshia, 2001b).

Researchers often only consider the effect of residual stresses at the part scale. Different approaches have been developed to simulate the residual stress at the part-scale level (Gouge et al., 2019; Setien et al., 2019; Tremsin et al., 2021). The inherent strain method induces strain in a small region of a part and applies it to the entire part to exclude the thermal simulation (Huang et al., 2016; Bugatti and Semeraro 2018; Chen et al., 2019). The “lumped” approach combines multiple layers of a build and applies a uniform heat source over the entire layer (Hodge et al., 2016; Yang et al., 2019). These methods allow prediction of part distortion with reasonable computation time, but lack resolution at the microscopic scale, which is required to study the effect of laser parameters and printing patterns on the LPBF process.

Residual stresses during LPBF are attributed to the large spatiotemporal thermal gradients from localized rapid heating and cooling (Attallah et al., 2016). To fully capture the effect of residual stresses, knowledge of the thermo-mechanical behavior of the material at different length scales: macro-stresses observed at the part scale level to micro-stresses at the grain dimension must be acquired (Attallah et al., 2016). However, beam-scale modeling is not feasible at the part scale due to the high computational costs, motivating the development of track-scale models (Irwin and Michaleris 2016; Luo and Zhao 2019; Huang et al., 2021; Tangestani et al., 2021). The first part of this series introduces a new track-scale model to account for the thermal behavior at the microscopic scale. The new Hybrid Line (HL) heat input model is derived from the 3D exponentially decaying (ED) heat input model from (Liu et al., 2018). The HL model accounts for the material state transition from powder to consolidated solid material and is calibrated for high gamma prime Ni-based superalloys by incorporating thermo-mechanical properties of the powder and fully dense material. The HL model increased the computational efficiency significantly (up to 1,500 times faster) compared to the ED beam-scale model. This track-scale simulation allows thermal behavior at the microscale to be applied on the part scale, enabling high accuracy and fast simulation times.

This second part of this series is devoted to the thermo-mechanical simulation of the LPBF process. Coupling the HL thermal model, developed in part I of this series, as input to a mechanical model, enables prediction of the residual stresses at the track and part scales. Firstly, a single-track simulation is applied to compare the residual stresses in the HL track-scale and ED beam-scale models. Secondly, part-scale simulations of thin-wall builds are completed using the HL track-scale models for comparison with experimentally measured residual stresses in LPBF parts. Specimens with different laser powers and printing patterns are used to evaluate the simulations. The computational efficiency of the thermomechanical model is further enhanced using a mesh coarsening technique in Abaqus.

2 MATERIAL AND EXPERIMENTAL METHODS

2.1 Material Composition

The material considered for LPBF is a gas-atomized high- γ' Ni-based superalloy RENÉ 65 (R65) powder produced by ATI Powder Metals, which predominantly consists of spherical particles ranging in size from 12 to 42 μm with a D50 size of 19 μm . The R65 chemical composition is 15% Cr, 13% Co, 4% W, 4% Mo, 3.5% Ti, 2.1% Al, 0.9% Fe, 0.7% Nb, 0.05% Zr, 0.04% Ta, 0.01% B, and the balance is Ni.

2.2 LPBF Procedure

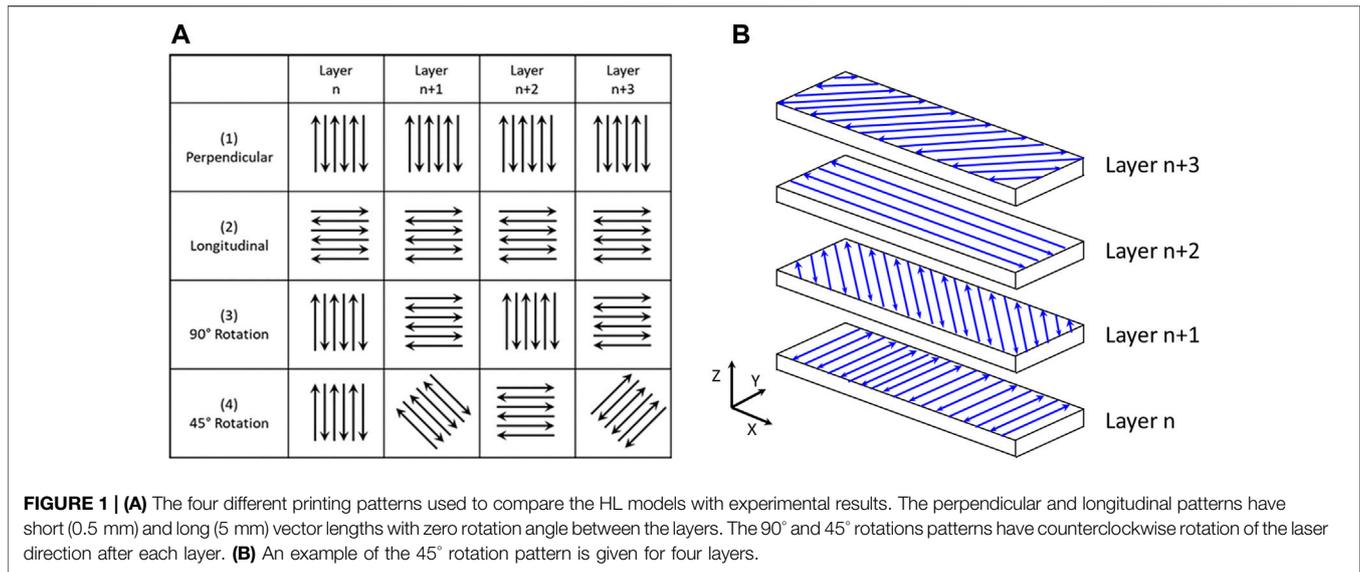
Twelve part-scale specimens are printed using an Aconity MIDI LPBF machine in an Argon environment to validate the thermo-mechanical HL model. The printed size of each specimen is $5 \times 1.2 \times 0.5$ mm (length \times height \times width) based on simulation time and X-Ray Diffraction (XRD) residual stress measurement considerations. Specimens are oriented at a 25° angle with respect to the recoater direction. Each component is spaced at least 10 mm apart to avoid negative impacts of adjacent laser processing. For this reason, the two nearest specimens were not printed subsequently. This eliminates concerns with the thermal effects of neighboring parts (Robinson et al., 2018; Scime and Beuth 2019).

The specimens are printed with a laser speed of 1,000 mm/s, layer thickness of 40 μm , laser radius of 60 μm and hatch spacing of 90 μm . A series of three different laser powers and four different laser scan path patterns are studied. The three different laser powers are, 180, 200, and 220 W, respectively. The four laser scanning patterns, referred to as longitudinal, perpendicular, 90° and 45° rotations are shown in **Figure 1A**. **Figure 1B** shows how the rotations are completed between the layers. Longitudinal and perpendicular scanning patterns have 0° rotations between the layers. This allows comparison of the vector length effect (longitudinal = 5 mm and perpendicular = 0.5 mm) on residual stresses. The effect of the rotation angle between layers on the residual stress is evaluated using the 90° and 45° rotation patterns.

2.3 Residual Stress Measurement

The residual stresses in the part-scale specimens are measured using a μ -X360s residual stress analyzer, manufactured by PULSTEC. The machine is equipped with a Cr K α source and a 0.3 mm collimator. The Young's Modulus and Poisson's Ratio are defined in the machine settings to calculate the residual stress from the measured strain. The cos α method (Taira et al., 1978; Delbergue et al., 2016) is used to measure the residual stress, where X-rays are 360° -omnidirectionally diffracted from the samples around the path of incident X-rays and are detected by two-dimensional detectors. The residual stresses were evaluated 3 times at the center of the specimen. The XRD scan time is about 10 min to allow approximately 500 measurements over the 360° debye ring.

Past studies have shown that surface roughness significantly affects the residual stress measurements (Serrano-Munoz et al.,



2021). Moreover, the residual stress of the final LPBF build layers is not essential as the free surface allows stress relaxation. It has also been found that the residual stresses are higher inside the LPBF parts (Robinson et al., 2018). For these reasons, the center section of the specimen is selected for residual stress measurement. Samples are mounted, ground, and polished using standard metallographic procedures. Approximately 0.6 mm (half of the sample height) is removed during sample preparation. To preserve the residual stresses, the samples are mounted while still attached to the base plate. The final polishing step is performed in a Beuhler Vibromet (TM) 2 Vibratory Polisher with 0.05-micron alumina solution (MasterPrep) for 72 h to ensure that the mechanical stresses induced by the mechanical polishing are minimized. While it is acknowledged that the removal of half the specimen affects the residual stress magnitudes, the differences in residual stresses between the different scanning patterns and laser powers are preserved.

3 MODELING

Two sequentially coupled thermo-mechanical models are implemented in Abaqus (a Dassault Systems finite element software) to evaluate the accuracy and computational time of the models. Firstly, a single-track model is implemented for comparison between HL and ED models introduced in Part I of this series (Tangestani et al., 2021). The beam-scale (ED) model is a baseline for comparison due to its greater accuracy in replication of the heat input profile of the energy source (Irwin and Michaleris 2016; Luo and Zhao 2019). Secondly, a HL-based part-scale model is developed for comparison with the experimental residual stress measurement described in *Residual Stress Measurement*.

The equations governing the thermal behavior of the ED and HL models has been previously described in Part I (Tangestani et al., 2021). Standard equations governing the mechanical

behavior have been used for both HL and ED models. Thermal gradients predicted by the thermal models are input into the mechanical models to predict the resulting stresses. The total strain increment $\Delta \varepsilon_{ij}$ is subdivided as per **Eq. 1**:

$$\Delta \varepsilon_{ij} = \Delta \varepsilon_{ij}^E + \Delta \varepsilon_{ij}^P + \Delta \varepsilon_{ij}^T + \Delta \varepsilon_{ij}^{\Delta V} + \Delta \varepsilon_{ij}^{Trp} \quad (1)$$

where ε_{ij}^E is the elastic strain, ε_{ij}^P represents the plastic strain, ε_{ij}^T is the thermal strain, $\varepsilon_{ij}^{\Delta V}$ is volumetric strain due to phase transformation, and ε_{ij}^{Trp} is transformation plasticity. In this study, $\varepsilon_{ij}^{\Delta V}$ and ε_{ij}^{Trp} are assumed to be zero, consistent with previous studies (Tawfik, Nasr, and El Gamal 2019). The thermal strain increment $\Delta \varepsilon_{ij}^T$ is obtained from thermal expansion relation in **Eq. 2** as follows:

$$\Delta \varepsilon_{ij}^T = \alpha \Delta T \delta_{ij} \quad (2)$$

Where α is the thermal expansion coefficient, ΔT is the temperature rise and δ_{ij} is the Kronecker delta function ($\delta_{ij} = 1$ if $i = j$, $\delta_{ij} = 0$ if $i \neq j$). The resulting stress increment is obtained from the elastic strain as follows:

$$\Delta \sigma_{ij}^E = D_{ijklm} \Delta \varepsilon_{lm}^E \quad (3)$$

Where D_{ijklm} is the fourth order isotropic elastic stiffness tensor calculated from Young's Modulus (E) and Poisson's Ratio (ν):

$$D_{ijklm} = \frac{E}{1 + \nu} \left[\frac{1}{2} (\delta_{ij} \delta_{lm} + \delta_{il} \delta_{jm}) + \frac{\nu}{1 - 2\nu} \delta_{ij} \delta_{lm} \right] \quad (4)$$

The temperature-dependent material properties of R65, including density, specific heat, latent heat, thermal conductivity, incorporated in the simulations are given in Part I (Tangestani et al., 2021). The HL model calibration coefficient ($C = 1.2$) is maintained from Part I of this series (Tangestani et al., 2021). A summary of the temperature-dependent thermal material properties is provided in **Table 1**. The Young's Modulus, Poisson's Ratio, plasticity, and thermal expansion

TABLE 1 | Temperature-dependent thermal material properties for R65.

Temperature (°C)	Thermal conductivity (W/(m.K))	Heat capacity (J/(kg.°C))	
		ED	HL
25	9.9	435	435
162.5	11.35	462	462
300	13.6	477	477
437.5	16.15	496	496
575	18.74	533	533
712.5	22.6	583	560
850	24.7	623	590
987.5	31.6	751	620
1,125	24.95	597	640
1,300	70	597	640

are taken from (Gabb et al., 2001). The Young's Modulus for the powder state of the material is assumed to be 1% of the fully dense value at room temperature, accounting for the negligible powder stiffness. A linear temperature-dependent piecewise stress-strain relationship accounts for the elasto-plastic behavior as has been done for similar materials in (Gabb et al., 2001).

3.1 Single-Track Models

The model geometry for the single-track simulations (HL and ED) is shown in **Figure 2**. The substrate is 2 mm long, 0.96 mm high, and 0.5 mm wide. A 0.04 mm thick powder layer corresponding to the experimental setup is added on top of the substrate. The powder and substrate are divided with the dashed blue line in **Figure 2**. The substrate is fixed to its surroundings to constrain movement in all directions. The laser path direction is aligned with the X-direction in **Figure 2**. The single-track simulations are completed with a laser power of 200 W and laser speed of 1,000 mm/s. The residual stresses are evaluated along the longitudinal and transverse directions, named lines 1 and 2 in **Figure 2**. Based on previous work (Tangestani et al., 2021), a value of $20 \left(\frac{W}{m^2 \cdot ^\circ C}\right)$ and 0.4 are used for the convection and radiation heat loss, respectively. The ambient and initial temperatures for the substrate are both set to room temperature, 25°C. DC3D8, and C3D8 elements in Abaqus are implemented for the thermal and mechanical models, respectively.

To determine the minimum mesh size required for the thermal and mechanical models a mesh sensitivity study is conducted. The minimum required mesh size from the sensitivity study is maintained for both the thermal and mechanical models. The region over which the laser passes is meshed with elements whose dimensions are 10 μm for both Y- and Z-directions, and 20 μm for the X-direction in the ED model. For the HL model, the laser-affected region is meshed with elements having dimensions of 20 μm in Y and Z, and 30 μm in the X-direction. In addition, coarser elements are employed for regions further from the laser heat source to decrease the computational time, as shown in **Figure 2**. The material state transition (solid, powder, and liquid) is incorporated within the model using the USDFLD subroutine described in Part I of this series (Tangestani et al., 2021).

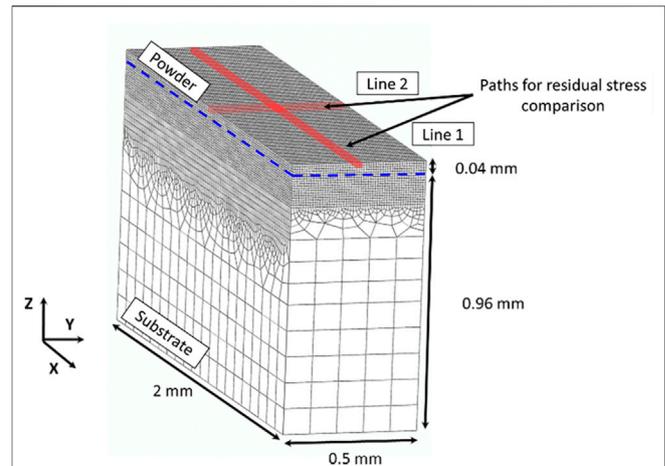


FIGURE 2 | Track-scale model mesh showing the dimensions of the substrate and powder bed. The laser movement is parallel to the X-direction. The residual stresses are evaluated along the two lines highlighted in red.

3.2 HL Part Scale Model

A sequentially coupled part-scale thermo-mechanical model with 30 build layers is developed for comparison with experimental results. The part is 5 mm long, 1.2 mm tall, and 0.5 mm wide and is built on a 4 mm thick substrate, as shown in **Figure 3**. A 0.04 mm thick powder layer corresponding to the experimental setup is applied during the simulation to mimic the printing process. The dashed blue line divides the printed part and the substrate in **Figure 3**. The substrate is constrained from moving in all directions while the part is printed.

Convective and radiative heat losses are defined for the free surfaces with the same coefficient values as the single-track model. The time between each layer addition is set to 10 s to reproduce the experimental settings for the recoater action. The simulation runs for 480 s after printing the last layer to allow the part to cool to room temperature. Following the build simulation, the boundary conditions are relaxed in the substrate to account for the stress relaxation when the part is removed from the printer.

The laser scan path is first generated within the Netfabb software and translated to a laser time-location database file using a custom Python script. The time-location file is then read by the UEXTERNALDB subroutine at the beginning of each increment in ABAQUS for the finite element simulation. The DFLUX subroutine uses this information to account for the position of the heat input during each time increment of the simulation.

A single τ value, representing the key control parameter for the HL model described in part I, cannot reproduce the different scanning patterns illustrated in **Figure 1** due to differences in laser path lengths. As a baseline from Part I, a maximum τ value of 10 is selected for speed and accuracy. For shorter path lengths, experienced in the perpendicular pattern or in the corner of the 45° rotation patterns, smaller τ values are applied to match the path length.

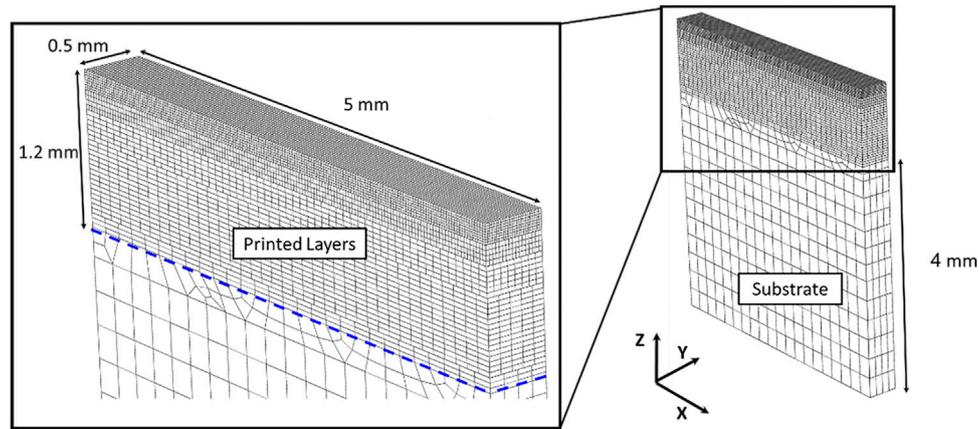


FIGURE 3 | Part-scale meshed 30-layer model used to simulate the 12 different LPBF builds (3 powers and 4 scanning patterns shown in **Figure 1**). The dashed blue line shows the border between the printed layers and the substrate. The residual stresses are taken along the line shown in red.

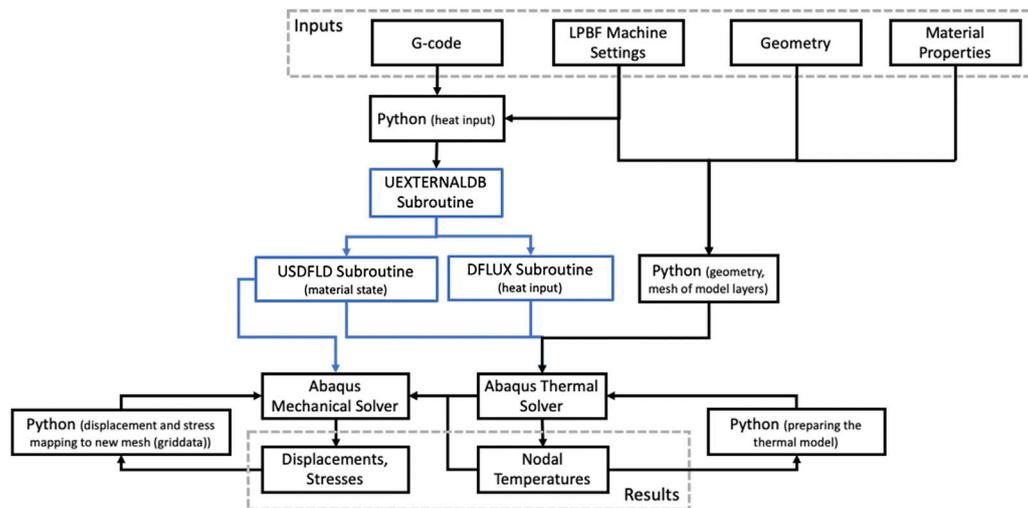


FIGURE 4 | Flowchart of the computational framework including subroutines (blue) and interactions with customized Python programs.

An adaptive mesh-coarsening technique is implemented to reduce the model size as each new layer is added and allows the thermal and mechanical models to run nearly simultaneously. A custom framework developed by Achuthan and Jayanath (Jayanath and Achuthan 2018) is implemented to coarsen the element size during simulation. The framework is incorporated with the element birth technique to simulate the layer deposition, and the phase transition is simulated with a user-defined subroutine. To facilitate the transfer of data from prior meshes, the stress-strain data is transferred using the “mesh-to-mesh solution mapping” feature within Abaqus. A Python script is developed to map the displacement from prior meshes to the new mesh using interpolation with the “griddata” tool (Jayanath and Achuthan 2018). Linear interpolation is adequate to calculate the updated position of the nodes, as

demonstrated in (Jayanath and Achuthan 2018; Hajjalizadeh and Ince 2019). A flowchart illustrating the computational framework is given in **Figure 4**.

Figure 5 shows the implementation of the framework to dynamically increase the element size during simulation. The build simulation begins with a single layer where the mesh is used to calculate the thermal results incorporated within the mechanical model. The nodal temperatures are used by the mechanical model to calculate the stresses and strains for each layer. Subsequent layers are meshed on top of the stack for further calculations using the results from the previous layers as initial conditions.

Table 2 indicates the mesh size changes as the number of layers is varied. This ensures high resolution close to the heat source and faster computation near the base plate. **Figure 5** shows how mesh coarsening is implemented after 5 and 30 build layers.

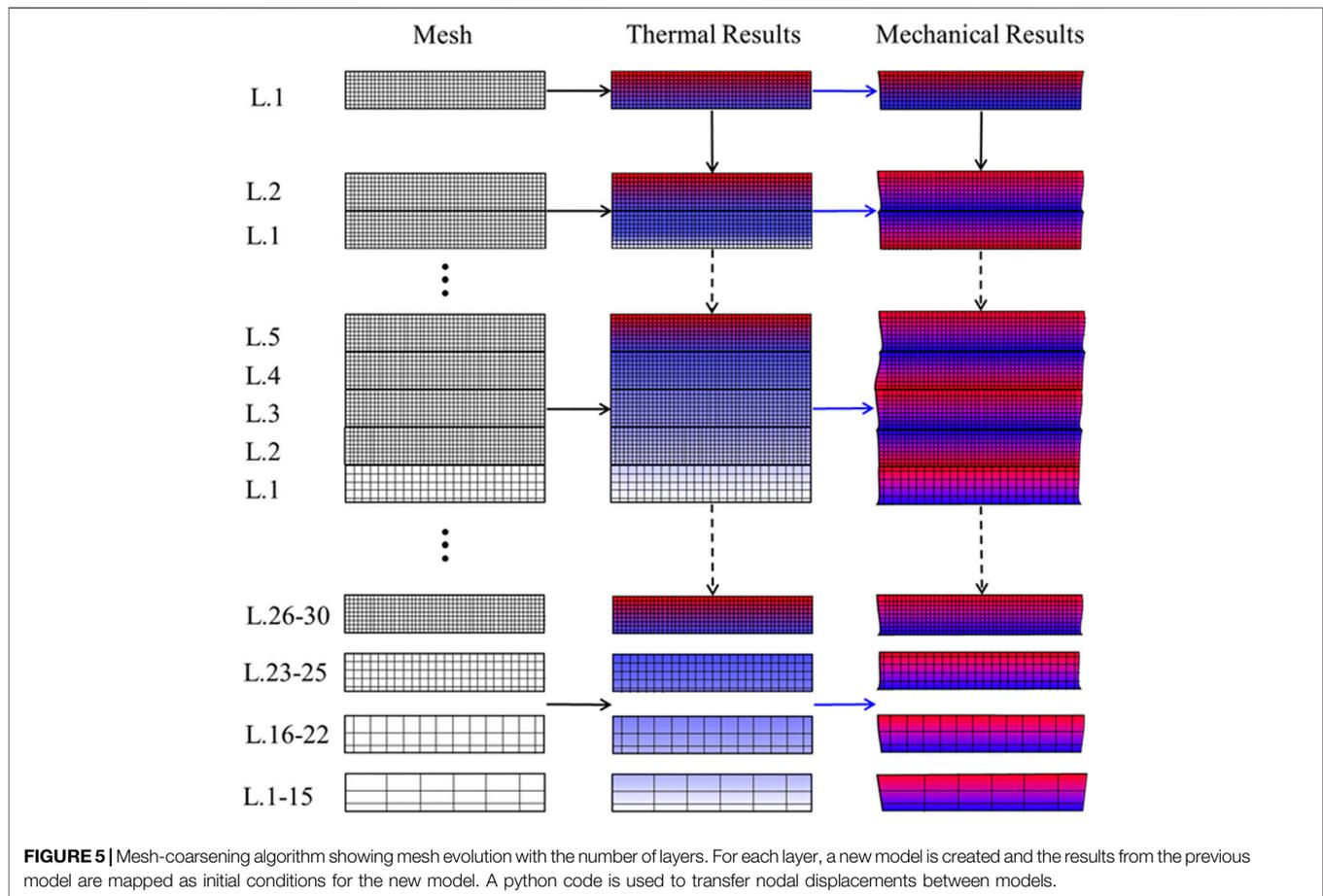


TABLE 2 | Element size, in microns, for each meshed layer of the part-scale simulation.

Layer's number	Length (X)	Width (Y)	Height (Z)
1–15	132	62.5	40
16–22	90	55	40
23–25	36	30	40
26–30	30	20	20

The mesh size of the first layer (L1 in **Figure 5**) is coarsened once the fifth layer (L5 in **Figure 5**) is deposited. As the build progresses, four different mesh sizes are incorporated, as detailed in **Table 2**. When the last layer is deposited, the layers L30 to L26 have the finest mesh size, while layers L1–L15 have the coarsest mesh size. The first 15 layers have 304 elements per layer and the last 5 layers have 4,175 elements per layer based on **Table 2**.

4 RESULTS AND DISCUSSION

4.1 Comparison Between HL and ED Single Track Models

The predicted residual stresses for the single-track model are examined to compare the HL and ED models. The results are

evaluated 10 s (layer-wise cooling time) after the track is printed with the boundary conditions relaxed.

4.1.1 Effect of Thermal Model on Stress Distribution

It was previously explained in Part I that the thermal distribution is crucial to accurately capture the stresses and strains generated during LPBF. This requires accurate prediction of nodal temperatures, cooling rates, and the heat transfer between the different material states (i.e., liquid, solid and powder states). The von Mises stress distributions are shown for the ED and HL models simulated with three different τ values in **Figure 6**. The stress is concentrated in the solidified regions (melted track and substrate). The powder bed has no stress (see the dark blue region in **Figure 6**) because it has negligible stiffness. Other researchers who have not considered the powder state have over-predicted stresses in the powder regions during LPBF (Irwin and Michaleris 2016). This demonstrates the importance of considering the material state while simulating the LPBF process.

The stress distributions are similar between HL $\tau = 5$ and the ED models. The stress-affected zones on the surface of the cross-sections have similar sizes and magnitudes. Both simulations have similar maximum stress amplitude situated at the center of the track. As τ increases, the stress-affected zone shrinks and the peak intensity decreases due to the heat source being integrated over a

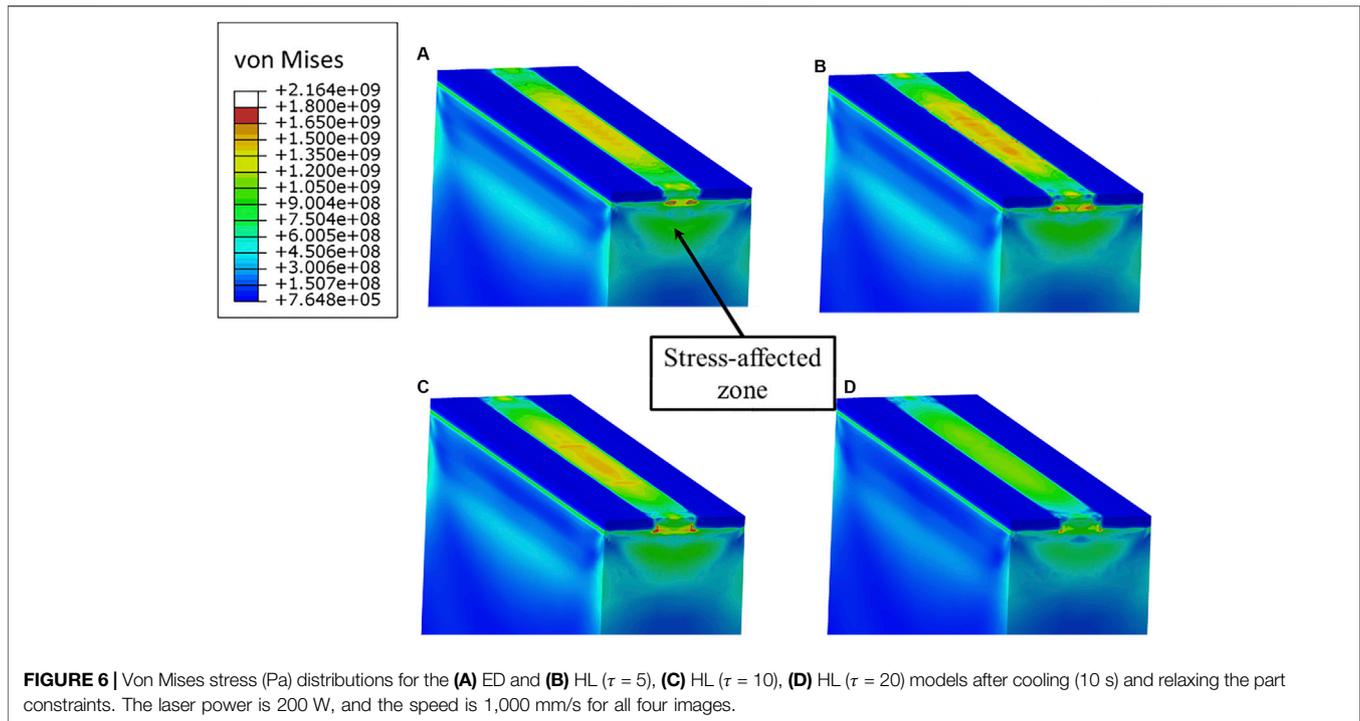


FIGURE 6 | Von Mises stress (Pa) distributions for the (A) ED and (B) HL ($\tau = 5$), (C) HL ($\tau = 10$), (D) HL ($\tau = 20$) models after cooling (10 s) and relaxing the part constraints. The laser power is 200 W, and the speed is 1,000 mm/s for all four images.

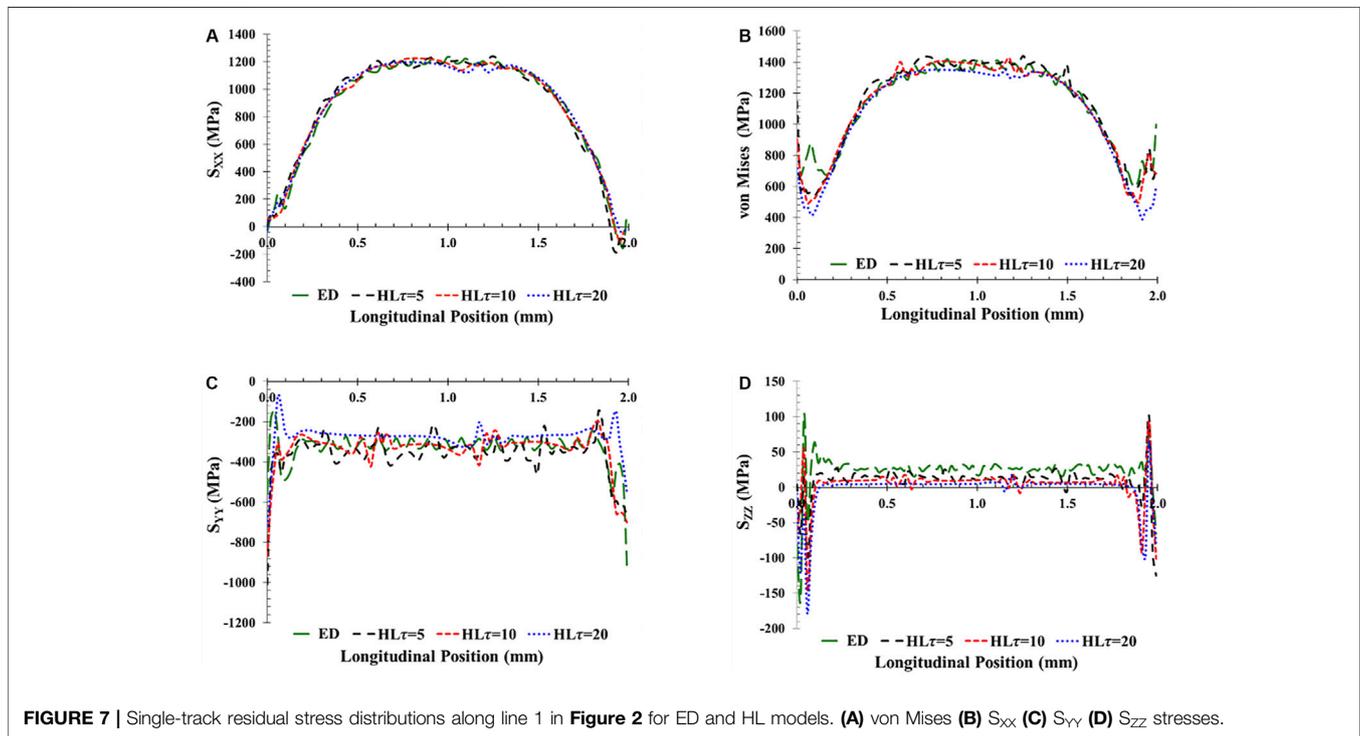


FIGURE 7 | Single-track residual stress distributions along line 1 in Figure 2 for ED and HL models. (A) von Mises (B) S_{xx} (C) S_{yy} (D) S_{zz} stresses.

longer distance, resulting in reduced peak temperatures. This effect could be reduced by calibrating the HL coefficients for every τ value. The implemented heat transfer coefficients are calibrated in Part I (Tangestani et al., 2021) by minimizing the error for four τ values ($\tau = 5, 10, 15,$ and 20).

4.1.2 Effect of τ on Residual Stresses

The distribution of the residual stress components, along with the von Mises stress for ED and three different τ values in the HL model, are shown in Figure 7, 8 for line 1 (longitudinal) and line 2 (transverse) in Figure 2, respectively. In general, the maximum

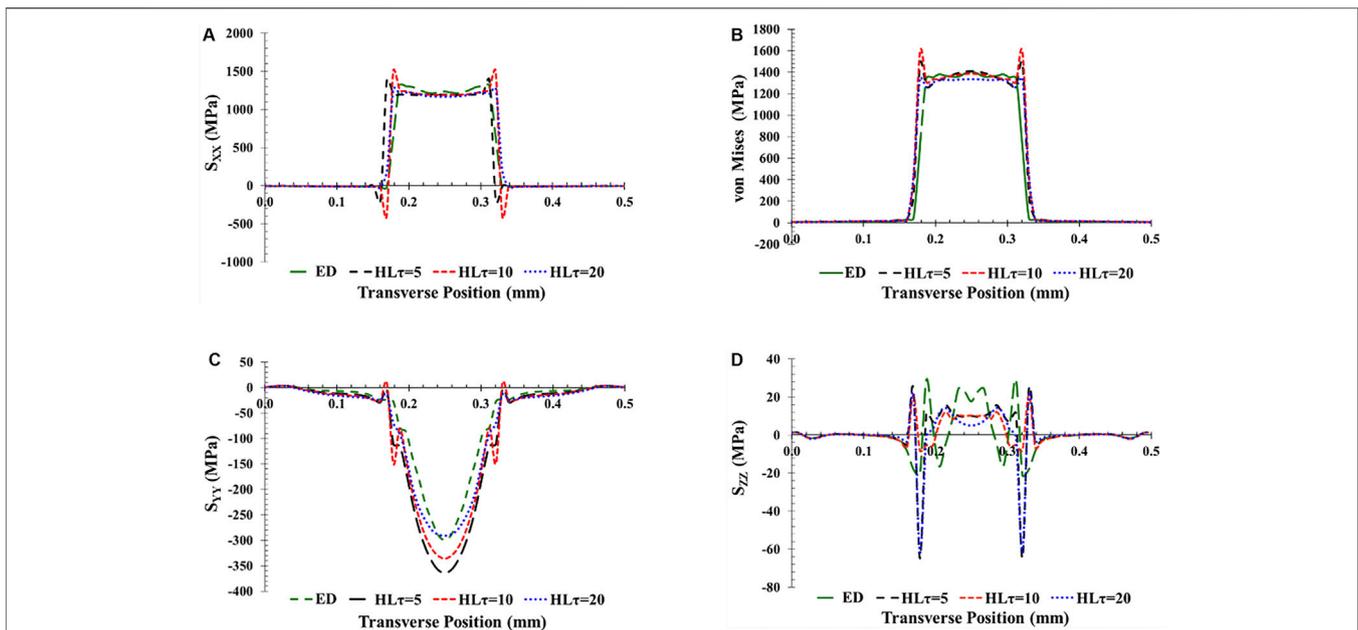


FIGURE 8 | Single-track residual stress distributions along line 2 in **Figure 2** for ED and HL models. **(A)** von Mises **(B)** S_{xx} **(C)** S_{yy} **(D)** S_{zz} stresses.

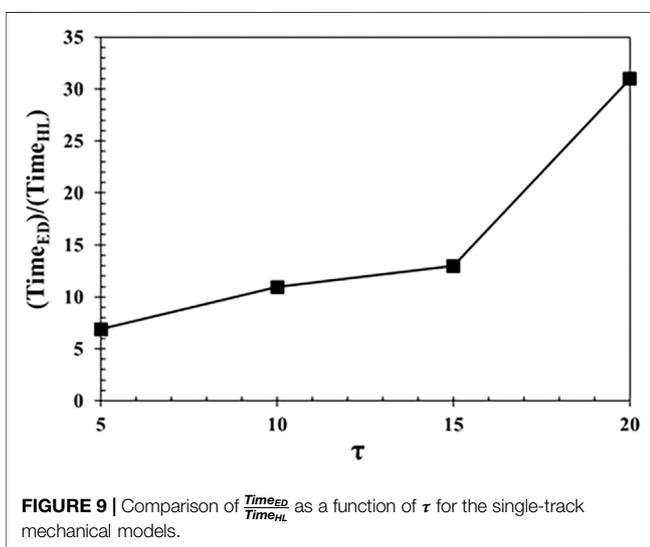


FIGURE 9 | Comparison of $\frac{\text{Time}_{ED}}{\text{Time}_{HL}}$ as a function of τ for the single-track mechanical models.

stresses are observed at the center of the track, where the expansion and contraction of the material are the most constrained by surrounding material. The stresses decrease to nearly zero close to the edges where the solidified material is free from constraints. The peak stresses are observed in the longitudinal direction (S_{xx} in **Figures 7, 8**). The stress magnitude is lowest in the build direction (S_{zz}) due to the free surface, consistent with previous LPBF single track simulations (Walker et al., 2019). The primary contribution to the von Mises stress comes from the X-direction stress. Stresses are mostly tensile in the X- and Z-directions, while they remain compressive in the Y-direction. The negative peak stress values

observed at the edges for the S_{yy} and S_{xx} in **Figure 7** are due to edge effects.

The HL model accurately reproduces the results of the ED model along the laser direction in **Figure 7**. The average variations between the ED and HL models are below 3% for all τ values in the S_{xx} stresses. As the τ value increases, the stress oscillation is lengthened because the heat input is distributed over a longer region. This reduces the peak temperature as discussed in Part I (Tangestani et al., 2021). Each stress oscillation represents one heat input step, and the larger the heat increment, the larger the variation in stress magnitude compared to the ED model. This is more noticeable in the Y- (mean variation = 12%) and Z- (mean variation 80%) directions when $\tau = 20$. Nevertheless, the Y- and Z-stresses have limited effect on the overall stress as shown by the accuracy in the von Mises stress (mean variation = 5%) distributions. Increasing the τ value has a similar effect on the variation of stress distribution calculated perpendicularly to the laser direction in **Figure 8**. On the contrary, the width of the stress distribution decreases with increasing τ due to the difference in melt pool size described in Part I (Tangestani et al., 2021).

4.2 Model Computational Efficiency

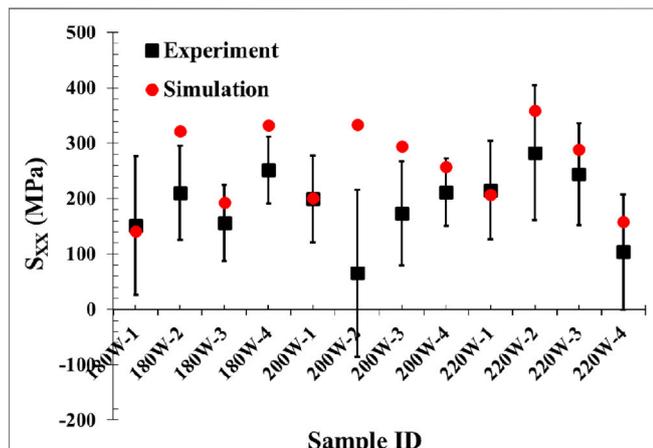
The computational efficiency of the HL model in comparison to the ED beam scale models is attributed to two factors. Firstly, the time step increment size (τ) contributes to speeding up both the thermal and mechanical calculations by reducing the number of calculations in the track-scale model for the given laser paths. Secondly, the larger τ value enables use of a coarser mesh size, reducing the data size for mesh-to-mesh mapping.

4.2.1 Effect of τ on the Computational Efficiency

Figure 9 shows the impact of τ on the computational efficiency of the mechanical model for the track-scale simulations using a

TABLE 3 | Computational time (hours) comparison between the ED beam-scale and HL ($\tau = 10$) part-scale models with and without the mesh coarsening technique.

	With mesh coarsening (hours)			Without mesh coarsening (hours)		
	Thermal Model (est.)	Mechanical Model (est.)	Total (est.)	Thermal Model (est.)	Mechanical Model (est.)	Total (est.)
ED	11,220	679	11,898	37,422	2,266	39,688
HL ($\tau = 10$)	17	62	62.5	56.7	207	264

**FIGURE 10** | Comparison between the measured and predicted residual stresses in the X-direction (S_{xx}) evaluated at the center of the specimens. For each specimen ID, the first number refers to the laser power and the second number represents the scanning strategy given in **Figure 1**.

consistent mesh. The HL model is over 6 times faster than the ED model when $\tau = 5$ and is over 30 times faster when $\tau = 20$. This shows that the time step increment has a bigger impact on the thermal model (1,500 times faster for $\tau = 20$ in (Tangestani et al., 2021)) compared to the mechanical model (30 times faster for $\tau = 20$), as the ED model requires more calculations for the thermal model. The time step increments thus have a smaller effect on the mechanical model. The total computational efficiency gain is the summation of the gain from the thermal and mechanical models for consistent meshes.

4.2.2 Effect of Mesh Size on the Computational Efficiency

Element size is another important factor when considering the computational efficiency of the model. It is desirable to have a small mesh size in regions where the laser interacts with the powder bed to increase the resolution of the model. However, decreasing the element size decreases the time step. Therefore, it is desirable to use the maximum possible element size for part-scale simulations and motivates the use of dynamic re-meshing.

Dynamic re-meshing is applied to the part-scale simulations as the printed layers are added. The ED model cannot be compared due to infeasible simulation time at the part scale. However, the estimated computational times for

the ED model in addition to the HL model, with and without dynamic re-meshing, are compared in **Table 3**. The estimated times are calculated by comparing the processing times of the ED and HL single track models for the thermal and mechanical simulations. The ratio of computation time is multiplied by the HL part-scale simulation time to estimate the time required for the ED model applied at the part scale. Dynamic re-meshing decreases the thermal and mechanical models computational time by 3.3 times, allowing the thermal and mechanical models to be run in parallel. Since the mechanical model is 3.6 times slower than the thermal model, it controls the total run time.

4.3 Part Scale Simulation of Residual Stresses

Twelve LPBF specimens with different laser processing conditions (3 different powers and 4 different scanning patterns shown in **Figure 1A**) are produced to evaluate the accuracy of the HL model at the part scale. The residual stresses along the longitudinal direction (S_{xx}) are measured at the center of the specimens as explained in *Residual Stress Measurement*. The simulated stresses (S_{xx}) are also evaluated at the centers of the specimens and are compared with the experimental values in **Figure 10**. Specimen IDs are given along the abscissa where the first number refers to the laser power (180–220 W) and the second number represents the scanning pattern, following the naming convention in **Figure 1**. The experimental error bars for each specimen corresponds with the standard deviation in residual stress measurement.

For most LPBF conditions, the simulated stress falls within the measured standard deviation. The maximum and minimum variations between the experimental and simulated S_{xx} stress components are 121 and 1 MPa, respectively. Most simulations overpredict the measured residual stresses. This is likely due to the stress relaxation associated with the metallographic preparation. The removal of materials has been shown in (Ding 2012; Robinson et al., 2018) to relieve the residual stresses. The specimen 200 W-2 shows larger deviation compared to the simulation (268 MPa). It is unlikely that the measured residual stress varies between tension and compression. This error is attributed to the difficulty of aligning the X-ray beam over the small sample area (2.5 mm^2). Acquiring residual stress measurements from multiple samples with the same printing conditions would provide additional statistical data and

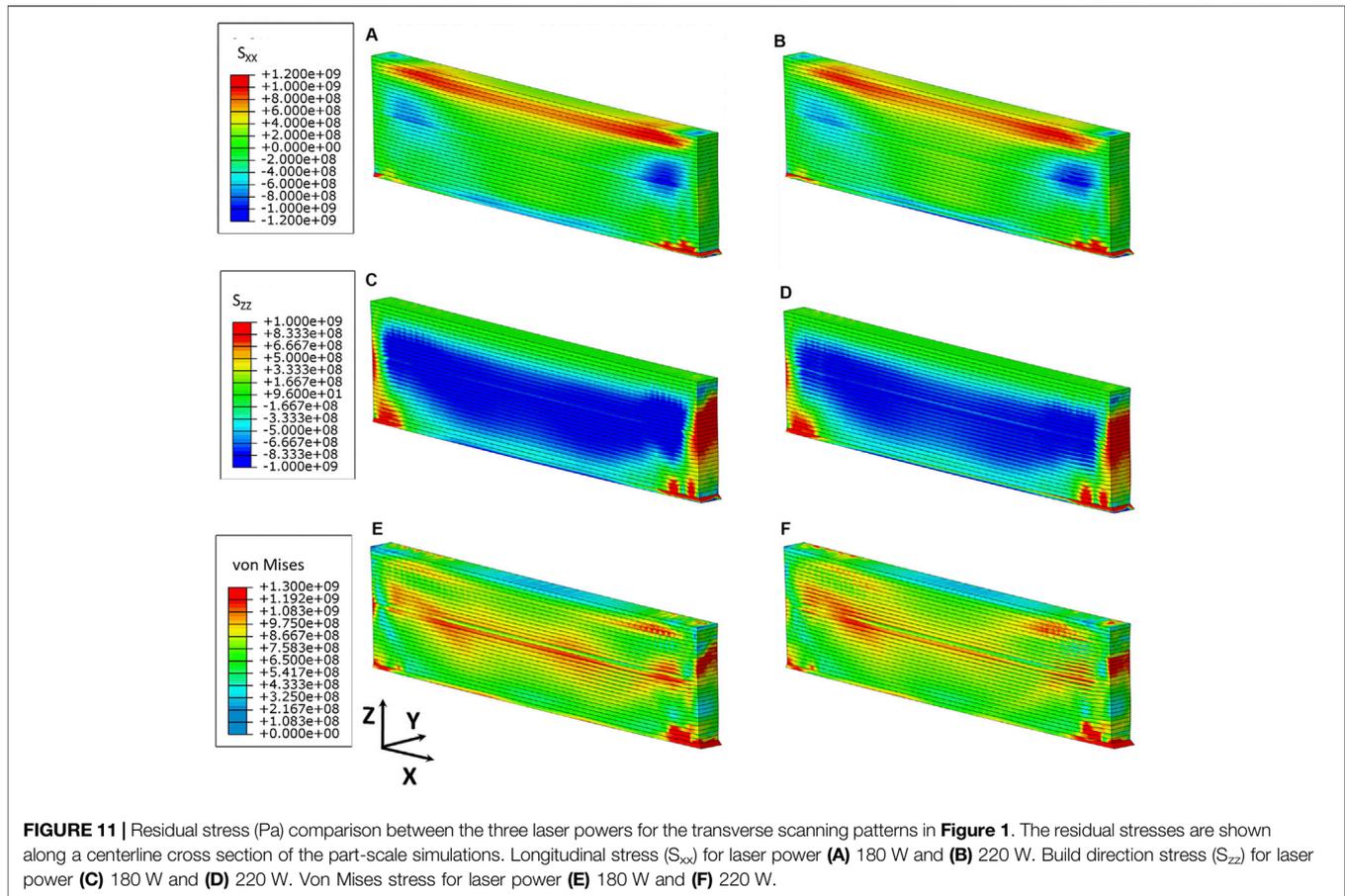


FIGURE 11 | Residual stress (Pa) comparison between the three laser powers for the transverse scanning patterns in **Figure 1**. The residual stresses are shown along a centerline cross section of the part-scale simulations. Longitudinal stress (S_{xx}) for laser power **(A)** 180 W and **(B)** 220 W. Build direction stress (S_{zz}) for laser power **(C)** 180 W and **(D)** 220 W. Von Mises stress for laser power **(E)** 180 W and **(F)** 220 W.

improve compatibility with the simulation results. Overall, the trends of increasing and decreasing the residual stresses with laser power and scanning patterns is well captured by the HL part-scale model.

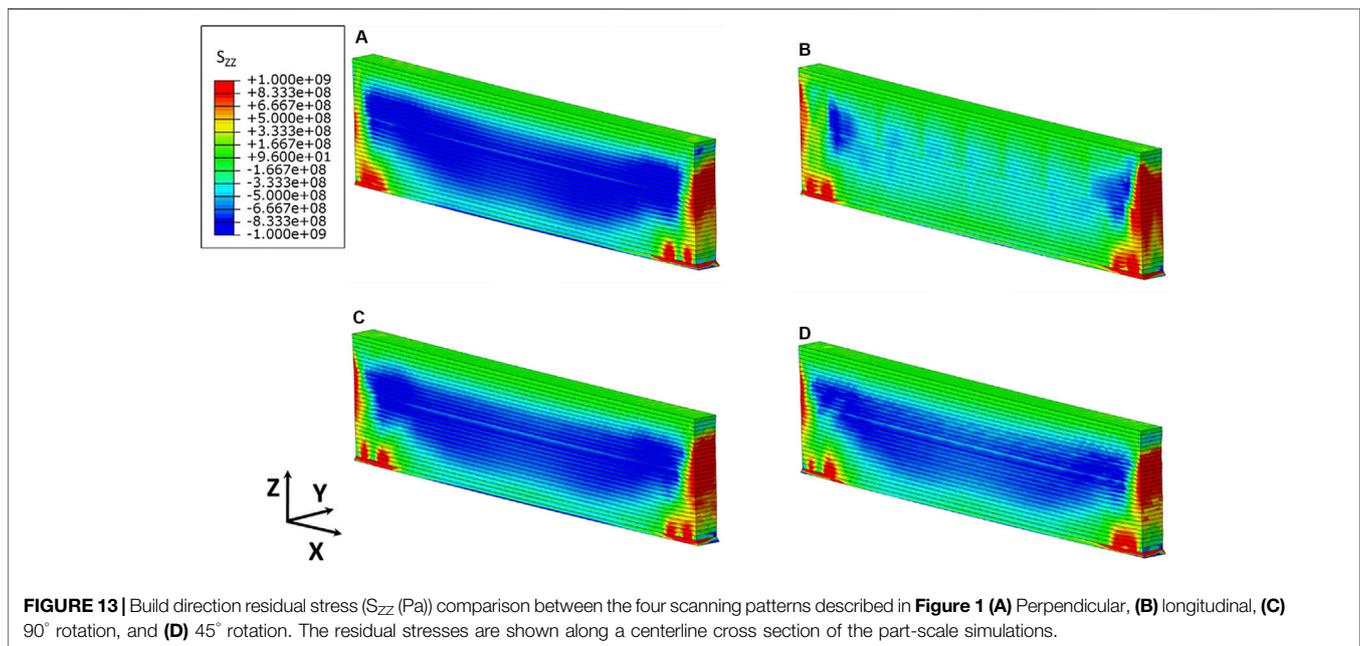
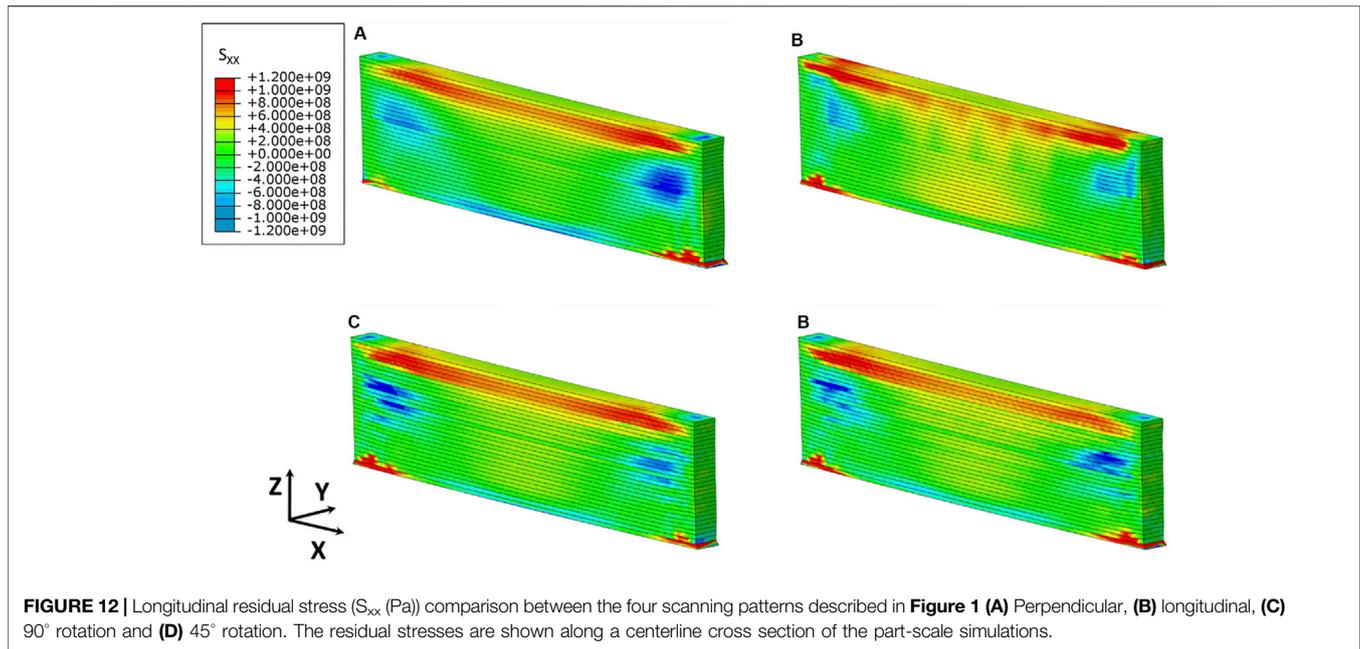
4.3.1 Effect of Laser Power on Residual Stresses

Currently, there is no consensus on the effect of laser power on residual stress. Some researchers have shown that laser power has little effect on residual stress (C. Chen et al., 2020), while others have demonstrated both positive and negative correlation on residual stress (Xiao et al., 2020). In Part I of this series, it is shown that the laser power has a strong effect on the melt pool size but a limited effect on the cooling rate. Therefore, it is expected that the laser power will have a small impact on the simulated residual stresses.

The simulated residual stresses obtained with laser powers of 180 and 220 W are shown in **Figure 11**. Only the perpendicular scanning patterns (see **Figure 1**) are shown here as the other scanning patterns exhibit similar results. The part-scale models are sectioned through the center to show the internal stresses. There is minute difference in the residual stress magnitudes and distributions for the different laser powers observed in **Figure 11**. It should be noted that the laser powers in these studies are limited to a small range. A larger laser power range could have a larger effect on the residual stresses.

The longitudinal stress component (S_{xx}) in **Figure 11A–B** is highest close to the top of the build. However, the S_{zz} and the von Mises stress in **Figure 11C–F** shows the largest stresses at the center of the part. The S_{zz} stress is near-zero at the top of the parts due to the free surface. The stress at the bottom of the part is reduced due to relaxation of the boundary conditions described in *HL Part Scale Model*. The S_{yy} stress is not presented because it does not change with laser scanning pattern and provides smaller contributions to the stress state. This is due to plate theory, which states that only limited load can be supported through the thickness direction in thin wall structures, as shown in (Chakraborty et al., 2021). Note that the inconsistency in the stress distribution around the center line is due to the mesh-to-mesh mapping, as shown in (Hajjalizadeh and Ince 2019).

The distributions of the cooling rates (not shown here) are similar for the three laser powers considered in this study. The maximum in-process cooling rate is 5.3×10^6 K/s for 220 W laser power and the cooling rate decreases to 4.3×10^6 K/s when the laser power is reduced to 180 W. These cooling rates are consistent with values commonly reported for LPBF processing (Wang et al., 2019). For such an infinitesimal variation in the cooling rate with laser power, the effect on residual stresses is expected to be

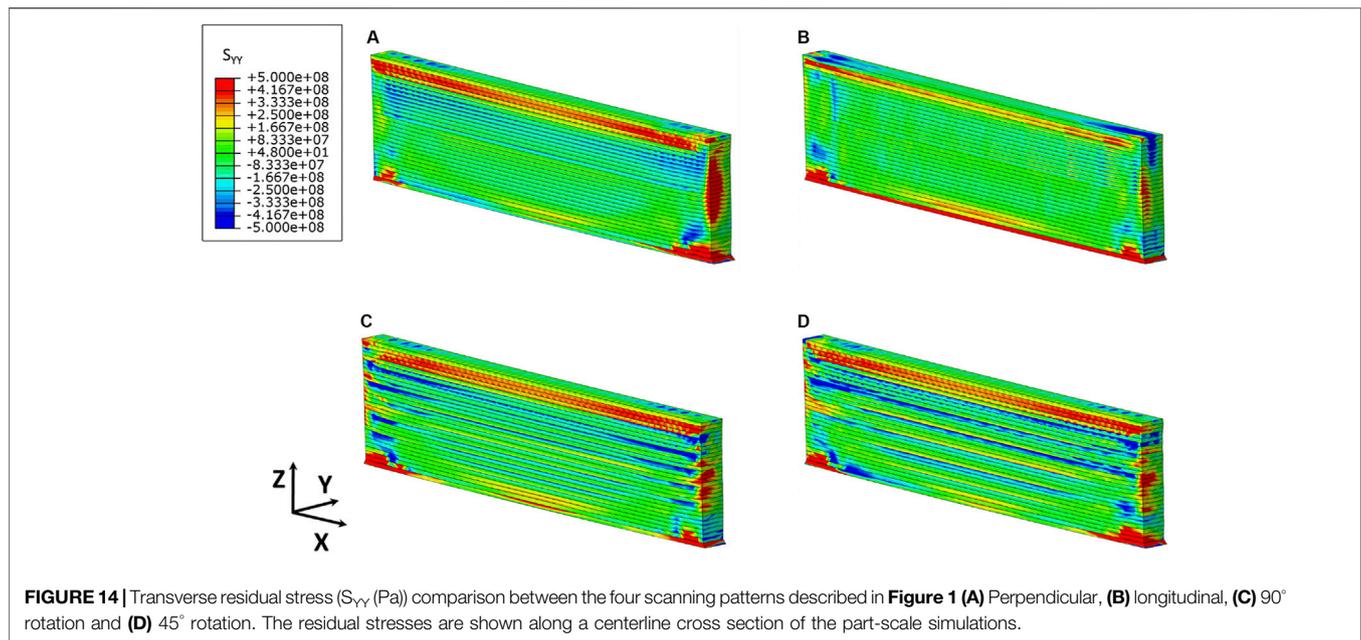


minimal. This is consistent with **Figure 11**, which shows minimal effect of laser power on the residual stresses.

4.3.2 Effect of Scanning Patterns on Residual Stresses

Laser printing patterns have been previously investigated by other researchers (Kruth et al., 2012; Setien et al., 2019; Serrano-Munoz et al., 2021). Generally, the scan strategy affects the final microstructure and part distortion more than the laser power (Xiao et al., 2020). Important parameters for investigating the effect of laser scanning strategy are the vector length (laser path

length) and the laser path rotation angle between the layers. Kruth *et al.* (Kruth et al., 2012) found that longer laser passes increase part distortion. Laser path rotation reduces the directionality of the residual stresses and creates a more homogenous quasi-isotropic stress state (Parry et al., 2016; Setien et al., 2019). However, these studies focused mainly on “island” scanning patterns for thick part components (Cheng and Chou 2015; Serrano-Munoz et al., 2021). To the authors knowledge, there have been no studies on the effect of scanning patterns on thin-wall structures.



This study focuses on thin-wall geometries where perpendicular and longitudinal scanning patterns have short and long vector lengths, respectively. Moreover, two rotation angles (90° and 45°) between the layers are investigated. The predicted residual stresses for the four different scanning strategies are shown in **Figures 12, 13, 14** for the longitudinal (S_{XX}), build (S_{ZZ}) and transverse (S_{YY}) directions, respectively. For each case, limits are placed on the contour plots to eliminate numerical artifacts from stress concentrations at the edge of the parts or due to mesh-to-mesh mapping (Hajjalizadeh and Ince 2019).

Maximum stresses are observed in the longitudinal direction (S_{XX}) for all scan strategies and have the same peak magnitude as the single-track simulations in **Figure 7A**. In the single-track simulation, it is observed that the stress along the build direction (S_{ZZ}) is negligible. However, for the part-scale simulation, there is an accumulation of S_{ZZ} stress at the center of the part height. This indicates the importance of considering both build geometry and height to compare residual stresses in LPBF. The S_{YY} stress plots in **Figure 14** shows layer-laminated structure for the 90° and 45° rotation patterns. This is due to the directional stresses and mismatches between the layers, resulting in a non-homogenous stress distribution as discussed in (Kruth et al., 2012; Parry et al., 2016; Serrano-Munoz et al., 2021). The S_{YY} component is significantly lower (>2X lower than S_{XX}) than the other directions due to the thin wall builds, as explained in *Effect of Laser Power on Residual Stresses*.

There is increased residual stress along the laser travel direction. This is shown in **Figure 12B**, where the laser runs parallel to the part, and in **Figure 14A**, where the laser travels perpendicular to the part. The long vector length generates more tensile residual stress (average $S_{XX} = 505$ Mpa) along the part length (**Figure 12B**), and the short vector length causes more residual stress (average $S_{YY} = 89.2$ Mpa) in the through-thickness

direction (**Figure 14A**). The resulting residual stress is proportional to the path length, as seen by comparing the peak stresses in the perpendicular and longitudinal laser passes. Therefore, aligning the laser path with the thickness will reduce tensile residual stresses in thin-wall parts, as shown by comparing **Figures 12A,B**.

The minimization of the tensile residual stress comes at the cost of increasing compressive residual stresses in the build direction, shown in **Figure 13**. The short vector length in **Figure 13A** drastically increases the compressive residual stress in the build direction compared to the long vector length in **Figure 13B**. The difference is approximately 3 times higher for the short vector length. This increase in compressive residual stress will negatively impact the limiting build height in thin-wall components, as observed in (Chakraborty et al., 2021).

The two rotation patterns shown in parts of **Figures 12C,D, 13C,D, 14C,D** have a layered quasi-isotropic residual stress distribution. This is consistent with previous studies showing more isotropic stress distribution when the laser path is rotated from layer to layer (Parry et al., 2016). It is more apparent in **Figures 12, 14** where the longitudinal and transverse stresses are dominant due to the variation from layer to layer in the sectioned plane (Z-X). The stress distribution for the rotation patterns is a combination of both perpendicular and longitudinal patterns, with the 45° rotation pattern exhibiting less variation between neighboring layers.

5 CONCLUSION

In the second part of this series, a new track-scale thermo-mechanical model is developed to predict the residual stress distribution during the LPBF process. The elasto-plastic properties of R65 are considered in the prediction of the

residual stresses. The simulation results are first compared with a beam-scale (ED) simulation of a single laser track. There is good agreement between the track-scale (HL) and beam-scale models for residual stress prediction within 3 and 5% variation on the S_{XX} and von Mises stresses.

The time step increment size of the mechanical model has a smaller effect on the computational time compared to the thermal model. The mechanical track-scale HL model is six times faster when $\tau = 5$ and 30 times faster when $\tau = 20$ relative to the beam-scale ED model. The dynamic mesh-coarsening algorithm improves the computational time by a factor of 3.3 by reducing the time associated with solving each increment, allowing parallel computation of the thermal and mechanical solutions.

The accuracy of the HL model is also evaluated with part-scale specimens. X-ray diffractions are completed to measure the S_{XX} stress components on specimens built with 3 different laser powers and 4 different scanning patterns. The measured stresses ranged between 282 and 65 MPa. The predicted stresses were within the standard deviation (average deviation of 54 MPa) of residual stresses for most cases. The simulation typically over-predicted the residual stresses due to the sample preparation procedure, however the trends match.

Laser powers between 180 and 220 W have minimal effect, whereas the scanning pattern leads to more variation in residual stress distribution. While long vector lengths result in more tensile stresses along the longitudinal direction, short vector lengths cause less tensile stresses due to the part thickness. However, this leads to more compressive residual stresses in the build direction. Laser rotation patterns lead to a preferential combination of properties from both short and long vector lengths. This study shows that the model is capable of accurately predicting the residual stress variation due to laser parameters and scanning strategies at the part scale.

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DATA AVAILABILITY STATEMENT

The original contributions presented in the study are included in the article/Supplementary Material, further inquiries can be directed to the corresponding author.

AUTHOR CONTRIBUTIONS

RT: Developed the thermo-mechanical model, evaluated the simulation results, measured the residual stress using XRD, and analyzed both simulation and experimental results along with writing the manuscript. TS: Contributed to the development of the modeling process and writing and editing the manuscript. AC: Prepared the samples for the residual stress experiment: mounting, grinding, and polishing; and reviewed the manuscript. LY: Printed the components for XRD measurements and reviewed model development. NK: Provided the RENÉ 65 powder for the LPBF machine, funded the project, edited the manuscript, and reviewed the XRD measurement process. ÉM: supervised the project and contributed to writing the manuscript.

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