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Thermo-economic assessment of metallic high-temperature latent heat storage system

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The promising prospects of high-temperature latent heat storage (HT-LHS) systems are accentuated by their advantages, including significant energy storage density, superior energetic efficiency, quasi-isothermal functionality, and seamless integration with renewable energy systems such as 3rd Gen Concentrated Solar Plant and Thermophotovoltaic systems. This study evaluates the thermo-economic performance of a proposed HT-LHS system having silicon as phase change material (PCM). A single thermal cell and a complete LHS system (consisting of several thermal cells) integrated with the supercritical CO₂ cycle are considered for the thermal and economic analyses, respectively. Furthermore, the charging performance of an equivalent thermal cell is compared with a specific Li-ion cell. Notably, a single thermal cell's gravimetric and volumetric energy densities surpass those of the specific Liion cell by approximately fourfold. Moreover, the charging time of the equivalent thermal cell, with minimal heat flow, is notably shorter than that of the Li-ion cell with comparable capacity. In terms of the levelized cost of electricity (LCoE), the HT-LHS technology demonstrates a significantly lower price of 9.547 Rs/kWh when storing 200 MWh of energy. Sensitivity analysis of LCoE reveals the opposite effect of loan repayment years (LOY) compared to other economic parameters. LCoE varies by 23.1%,16.43%,14.4%, and 8.06% by changing Return on equity (ROE), interest rate on the loan (IOL), Operation and maintenance cost, and discount rate from -40% to 40%.

KEYWORDS

high-temperature, latent heat storage, Li-ion battery, thermo-economic, LCOE

1 Introduction

Effective energy storage is critical in addressing the discrepancy between energy supply and demand and enhancing the dispatchability of renewable energy. Energy can be stored in different forms, such as Mechanical, thermal, electrical, electrochemical, etc.(Nadeem et al., 2019; Huang et al., 2015). Electrochemical storage, especially Li-ion battery storage technologies, is currently considered the most popular technology for storing electrical energy. Li-ion battery storage system has benefits of high energy density, long cycle life, low self-discharge rate, and low maintenance requirements. As demand for portable electronics and electric vehicles grows, Li-ion batteries will likely continue to play an essential role in the energy storage market (Goodenough, 2014).

Unlike Li-ion storage technology, thermal energy storage (TES) can store thermal energy in sensible, latent, or chemical forms (Alva et al., 2018). Thermal storage systems

operating at high-temperature are becoming increasingly appealing and effective methods to integrate with 3rd Gen concentrated solar plant (CSP) systems to generate combined heat and power effectively and efficiently. Integration of HT-TES systems can effectively reduce the temporal discrepancy between energy supply and demand caused by solar energy intermittency. Thus, maximizing the reliability and dispatchability of solar energy (Ray et al., 2023). Hence, HT-TES systems represent a potential approach to realizing sustainable energy growth and addressing the dynamic energy needs. Among the thermal storage technologies, HT-LHS and TIPV systems can be effectively integrated to produce power through direct conversion of heat (Datas et al., 2018a). Moreover, LHS has higher energy storage density than sensible heat storage (SHS) and higher maturity than thermochemical heat storage (TCHS) (Nazir et al., 2019).

Numerous investigations have concentrated on creating, conceiving, and assessing LHS systems that employ diverse PCM for gathering thermal energy. By selecting an appropriate phase change material and operating temperature, an LHS system can be tailored to suit a specific application (Aftab et al., 2021). LHS technology has been successfully employed for low (0°C-100°C) and medium (100°C-250°C) temperature applications. However, the technology needs to be evolved at higher operating temperatures to harness maximum benefits. Opolot et al. reported the key parameters affecting the performance of the LHS system operating above 500°C (Opolot et al., 2022).

Third Generation CSP systems with central receivers can attain temperatures exceeding 600°C, enabling their integration with highperforming power conversion systems like the supercritical carbon dioxide (sCO₂) Brayton cycle. This elevated temperatures compared to traditional Rankine cycles enhance the solar to power conversion efficiency, reducing the levelized cost of electricity (LCOE) (Mohan et al., 2018; Ramos et al., 2022). Suitable PCM is essential for hightemperature latent heat storage (HT-LHS) to maximize advanced power conversion benefits. Additionally, cogeneration of heat and electricity is efficient at temperatures above 900°C (Luft, 1985), and temperatures exceeding 1,000°C can be directly converted to electrical energy using Thermionic photovoltaic (TIPV) converters (Datas, 2016). Thus, coupling HT-LHS stand-alone systems with TIPV devices enables generation of power directly from high-temperature thermal energy.

HT-LHS systems utilizing metallic PCMs can achieve superior energy densities and storage rates compared to systems having inorganic salts and thus at elevated temperatures enhance exergetic and energetic efficiency of the system (Chen, 2015). Kim et al. presented a systematic process for selecting eutectic inorganic salts for use in latent heat thermal energy storage (LHTES) in the temperature range 450°C-500°C (Kim et al., 2024). Metallic silicon (melting point 1,414°C) outperforms traditional inorganic salt high-temperature PCM, boasting higher thermal conductivity (25-50 W/m K) and energy density (Datas et al., 2018b). Ray et al. conducted a numerical analysis to examine the thermohydraulics of silicon melting in a rectangular cavity and compared it with sodium nitrate (NaNO₃) (Ray et al., 2021). Zeneli et al. explored how porosity, vessel configuration, working parameters, and Stefan number affect HT-LHS charging rate using silicon (Zeneli et al., 2019). Hosseini et al. conducted a combined experimental and numerical study to understand the role of buoyancy-driven convection during constrained melting of phase change materials (PCMs) inside a shell and tube heat exchanger. They found that increasing inlet heat transfer fluid (water) temperature to 80°C reduced PCM melting by 37% in the shell and tube heat exchanger (Hosseini et al., 2012). The first largescale HT-LHS prototype, utilizing silicon as PCM, was introduced by Climate Change Technology in Australia in early April 2019 (Technology, 2019).

Energy storage systems can be assessed for their technical and economic viability through techno-economic analysis which combines thermodynamic and economic models to identify the optimal trade-offs between energy efficiency and economic performance, leading to cost savings and improved sustainability (Balli and Caliskan, 2022). Techno-economics is increasingly relevant for transitioning to a low-carbon economy, offering a holistic view of technology costs and benefits to inform policy and industry decisions (Mohammadi et al., 2021). Emrani et al. provided an overview of recent developments in the field of energy storage; combining a comprehensive assessment of the technical and economic characteristics of the various types of energy storage systems (Emrani and Berrada, 2024). Shan et al. proposed a novel layout integrating LHTES with a heat pump and assessed it according to different seasons, LHTES height-to-diameter (H/D) ratios, mass ratios of inflow water to radiator return water, and levelized cost of energy (LCOE) (Shan et al., 2024). Jorgenson et al. found that CSP-TES setups have lower overall costs than PV with batteries, even with the least expensive battery cost estimation (Jorgenson et al., 2016). Liu et al. conducted a techno-economic performance comparison between sensible and latent heat storage for CSP plants, highlighting impact of geometric dimensions and cost hypothesis on techno-economic performance of the TES system and suggesting that low thermal efficiency storage system can be cost-effective (Liu et al., 2021). Musi et al. performed a technoeconomic analysis of CSP system using LCoE as economic indicator (Musi et al., 2017). However, more literature on techno-economic analysis for LHS systems is needed, creating a knowledge gap regarding their economic viability and thermodynamic performance. Further techno-economic feasibility research is needed to explore the feasibility of using LHS for various applications.

Thermo-economic analysis of HT-LHS systems can provide valuable information from thermodynamic, economic, and environmental perspectives. Comparing the performance of thermal energy storage with battery energy storage can offer insights of their potential advantages and drawbacks, informing future energy storage decisions, and paving the way for further research. In this context, the objectives of the current study are:

- 1. Concept of integration of a standalone HT-LHS system having silicon as PCM with a supercritical CO₂ cycle for power generation.
- 2. To elucidate a novel approach for comparing the technoeconomic performance of the proposed HT-LHS system with a conventional Li-ion battery energy storage system (BESS) of the same capacity.
- 3. To estimate the levelized cost of electricity for both storage technologies for a rated power output of 50 MW and an energy capacity of 200 MWh.



FIGURE 1

Layout of two proposed strategies of integration for HT-LHS. (a) 3rd Gen CSP + HT-LHS + sCO₂ cycle and (b) Spillage from wind and PV plant + HT-LHS + TIPV/sCO₂ cycle.

4. To perform a sensitivity analysis of LCoE of HT-LHS system

2 Proposed standalone hightemperature LHS system

The HT-LHS system combines two technologies for simultaneous heat and electricity production: a high-temperature latent system with third-gen CSP using a sCO₂ cycle for solar-heat-power conversion and integrating a TIPV system with wind or PV power plants for power-heat-power conversion. Figure 1 illustrates the schematic layout of both strategies to generate heat and power.

The sCO₂ recompression power cycle employs sCO₂ as its working fluid, converting thermal energy into mechanical energy for electricity generation. This innovative technology offers advantages over traditional steam cycles, such as higher efficiency, reduced environmental impact, and smaller equipment size. The cycle includes components like a gas turbine, main compressor, recompressors, recuperators, heater, and condenser, with the HT-LHS system serving as the heater. In a standalone setup, the HT-LHS system accumulates thermal energy from renewable sources like solar, photovoltaic, or wind power, storing it in the proposed metallic silicon PCM that has a melting point of 1,414°C. The stored thermal energy is transferred to the sCO₂ to raise its turbine inlet temperature, improving cycle efficiency.

Thermionic photovoltaic (TIPV) is a solid-state device that directly converts thermal energy into electricity, merging thermionic and thermophotovoltaic units without intermediate stages. TIPV emits electrons from a cathode to a nearby cold anode in a vacuum within a range of 0.1–100 μ m, effectively converting heat into electrical energy. It combines electrons and

photons for enhanced power density and heat transfer. It is compatible with different heat sources, including solar concentrators, waste heat, and phase change materials. TIPV is particularly effective with silicon-based HT-LHS technology, which operates above 1,000°C.

3 Methodology

This section compares and discusses the thermodynamic and economic performance of the proposed standalone HT-LHS system with that of the Li-ion battery energy storage system (BESS). Electrical energy spillage from wind and PV power plants can be harnessed and stored as electrical and thermal energy using Li-ion batteries and HT-LHS systems, respectively. From a thermodynamic perspective, electrical energy is considered as high-grade energy and can only be converted to equivalent thermal energy at high temperatures. In this study, the author simulated energy spillage using electrical energy as input to both storage technologies.

The HT-LHS system is illustrated as a thermal battery or cell. A single cell with equal capacity is being evaluated for thermal and Liion cell to compare their thermal performance. The thermal cell in this comparison is a single pass, multi tube phase change heat exchanger that utilizes silicon as PCM. In order to perform an economic comparison between thermal and Li-ion battery, a system with high capacity is being evaluated. Henceforth, the HT-LHS system is named as thermal cell or battery in the remaining section of this article. Figure 2a represents a flowchart illustrating electrical energy storage using both technologies. A detailed methodology for techno-economic comparison is shown in Figure 2b.



4 Thermal performance comparison

Thermal performance is compared between a Li-ion cell and an equivalent thermal cell. A commercially available cylindrical Li-ion cell (LG 18650HG2) is considered for the analysis and its equivalent thermal cell is designed using thermodynamic principles. The technical specification of the Li-ion cell is mentioned in Table 1. Thermal analysis of Li-ion cell is performed considering the analogy between electrical with thermal circuits as shown in Table 2.

The storage capacity of the selected Li-ion cell is 10.8 Whe (38.88 kJ) as evaluated from the specification sheet. In order to determine the thermal energy equivalent of the stored electrical energy, the cycle efficiency of the sCO_2 cycle is considered and evaluated to be 60.1%. It is estimated that the equivalent thermal energy to be stored in the thermal cell is 64.8 kJ, and the design of the thermal cell utilizes metallic silicon as the high-temperature PCM. The mass and volume of silicon required for thermal cell is estimated utilizing the equation "Storage capacity = mass × latent heat". According to the calculation, the estimated mass and volume of silicon needed are 0.02 kg and 0.0078 ltr respectively, considering the

density of silicon ($\rho_{silicon}$) = 2,330 kg/m³ and a latent heat of fusion of silicon ($h_{sl,silicon}$) = 1,800 kJ/kg.

A multi-tube single-pass high-temperature phase change system is proposed to develop the thermal cell. The dimensions of the thermal cell are estimated based on the volume of the cell, with a length of 140 mm, shell diameter of 28 mm, and tube diameter of 12 mm. Similar to the constant current charging of Li-ion cells, the thermal cell is charged by subjecting the outer wall to a uniform heat flux without the flow of sCO₂. In contrast, discharging occurs as cold sCO₂ flows inside the tube and absorbs energy from the molten PCM. Meanwhile, the 3D model of the lithium-ion cell uses the actual dimensions of the cell, with a diameter of 18 mm and height of 65 mm. For a visual representation of the dimensional parameters of the two systems, refer to Figure 3.

4.1 Numerical formulation

ANSYS Fluent software is utilized to model the charging or melting of silicon in the thermal cell by employing the fixed grid TABLE 1 Specification of the selected Li-ion cell (LG, 2014).

Туре		Specification	Actual		
Chemistry		Li [NiMnCo]O ₂ (H-NMC)/Graphite + SiO			
Dimensions (mm)	Diameter	18.3 + 0.2/-0.3 mm			
	Height	65.00 ± 0.2 mm			
Weight (g)		Max. 48 44 ~ 45			
Nominal voltage (V)		3.6			
Charge method		Nominal: 1.5 A, 4.2 V, 50 mA End current (CC-CV)			
		Fast: 4 A, 4.2 V, 100 mA End current (CC-CV)			
Charge time	time Nominal (min) 165 min				
	Fast (min)	85 min			
Charge current	Nominal current (A)	1.25 A			
	Max. Current (A)	4 A			
Discharge End voltage (V)		2V			
	Max. Current (A)	20 A			
0.2C Capacity	Nominal (Ah)	3.0 Ah			
Energy density	Nominal (Wh/kg)	240			

TABLE 2 Thermal-electrical circuit analogy.

HT-LHS	Enthalpy (J)	Heat flow (W)	Temperature (K)	Thermal capacity (J/K)
Li-ion battery	Charge (Coulomb)	Current (Ampere)	Voltage (Volt)	Capacitance (Faraday)

enthalpy-porosity methodology through the Finite volume approach. While charging, the solid-liquid interface is treated as a mushy region that operates as a pseudo-porous zone, exhibiting different levels of porosity between 0 and 1. In this given scenario, when the porosity is 0, it signifies the solid phase, and when it is 1, it indicates the liquid phase. The equations used to conserve energy, mass, and momentum utilized in constructing the model are presented below (Seddegh et al., 2015). The energy equation can be expressed in terms of total enthalpy as Equation 1. Total enthalpy can be expressed in terms of sensible enthalpy and melting fraction in Equation 2. Sensible enthalpy can be expressed as a function of specific heat and reference enthalpy at reference temperature in Equation 3. The energy equation is solved numerically to obtain the temperature distribution in the domain.

$$\frac{\partial(\rho H)}{\partial t} + \nabla . (\rho \nu H) = \nabla . (k \nabla T) + S$$
(1)

$$H = h + \beta h_{sl}$$
 (2)

$$\mathbf{h} = \mathbf{h}_{\text{ref}} + \int_{T_{ref}}^{T} c_p dT \tag{3}$$

Where h is sensible enthalpy, β is liquid/melting fraction, S is energy dissipation, H is total enthalpy, h_{ref} is sensible enthalpy at T_{ref} and h_{sl} is the latent heat of fusion. The liquid fraction can be defined as in Equation 4. Solving Equation 4 numerically results in the fraction of molten PCM in the complete PCM domain.

$$\beta (T) = \text{liquid fraction} = \begin{cases} 0 \quad T < T_{sol} \\ \frac{T - T_{sol}}{T_{Liq} - T_{sol}} \quad T_{sol} \le T \le T_{liq} \\ 1 \quad T > T_{liq} \end{cases}$$
(4)

Replacing β (T) in Equation 1, the energy equation boils down to Equation 5:

$$\frac{\partial(\rho h)}{\partial t} + \nabla .(\rho v h) = \nabla .(k \nabla T) - \frac{\partial(\rho \beta h_{sl})}{\partial t} - \nabla .(\rho v \beta h_{sl}) + S$$
(5)

Where T is instantaneous temperature, T_{sol} and T_{liq} are solidus and liquidus temperature of PCM, respectively. The range of values for the parameter β determines the physical state of a region: $\beta = 0$ denotes a solid region, $\beta =$ 1 represents a liquid region, and $0 < \beta < 1$ indicates a mushy zone that contains both solid and liquid phases. The Navier-Stokes equation, which accounts for natural convection, can be written as Equation 6, and porosity function can be expressed as Equation 7. The momentum equation results in velocity distribution in the PCM domain as a function of melting fraction. After PCM begins to melt, there will be velocity of molecules in liquid PCM.

$$\frac{\partial(\rho v)}{\partial t} + \nabla . (\rho v v) = -\nabla p + \nabla . (\mu \nabla v) + \rho g + A v \tag{6}$$



$$A(\beta) = \frac{(1-\beta)^2}{\beta^3 + \epsilon} C$$
(7)

In the context of PCM, function A represents porosity. In contrast, the constant C represents the characteristics of the mushy zone that arises during the temperature interval between the solidus and liquidus points of the material. The morphology of the mushy zone influences the value of C. To prevent division by zero, a small constant (ϵ) value (0.001) is added to the denominator of the expression. The porosity function allows the momentum equation to mimic the Carman-Kozney equations for flow in porous media. Mushy zone constant measures the amplitude of velocity damping. Very large values of the mushy zone constant may cause the divergence in simulation. A value of 10^5 is considered in this study.

Except for the buoyancy force that generates natural convection, Boussinesq approximation considers the constant fluid density in all terms of the momentum equation. Reference density (ρ_{ref}) and reference temperature (T_{ref}) are used to model the body force. As a result, the momentum equation boils down to Equation 8. Density variation can be considered to follow bousinessq approximation as in Equation 9. The Bousinessq approximation considers linear variation of density with temperature.

$$\frac{\partial \rho_{ref} \nu}{\partial t} + \nabla . \left(\rho_{ref} \nu \nu \right) = -\nabla p + \nabla . \left(\mu \nabla \nu \right) + \left(\rho - \rho_{ref} \right) g + \frac{\left(1 - \beta \right)^2}{\beta^3 + \epsilon} C \nu$$
(8)
(9)

$$\left(\rho - \rho_{ref}\right)g = -\rho_{ref}k_T\left(T - T_{ref}\right) \tag{9}$$

The continuity equation can be expressed as Equation 10.

$$\frac{\partial \rho}{\partial t} + \nabla . \left(\rho \vec{V} \right) = 0 \tag{10}$$

The thermal and electrochemical characteristics of Lithium-ion cells are analyzed using Ansys Fluent. The battery model is employed to perform a thermal analysis of a Lithium-ion battery. The rate of heat generation within the battery is determined through a simulation that couples thermal and electrochemical effects. Within the battery cell, the transfer of lithium ions across the anode-separator-cathode sandwich layers is the primary physical process. The Multi-Scale Multi-Domain (MSMD) approach simulates the different physical processes happening in the active zone of the Li-ion battery. This approach involves solving the differential equations governing the thermal and electrical fields of the battery system domain as written from Equations 11–15; (Suresh Patil et al., 2021):

$$\frac{\partial \rho C_p T}{\partial t} - \nabla \cdot (k \nabla T) = \dot{q}$$
(11)

$$\dot{q} = \sigma_+ \left| \nabla \varphi_+ \right|^2 + \sigma_- \left| \nabla \varphi_- \right|^2 + \dot{q}_{ECh}$$
(12)

$$\dot{q}_{ECh} = j_{ECh} \left[U - \left(\varphi_+ - \varphi_- \right) - T \frac{dU}{dT} \right]$$
(13)

$$\nabla \cdot (\sigma_+ \nabla \varphi_+) = -j_{ECh} \tag{14}$$

$$\nabla \cdot (\sigma_{-} \nabla \varphi_{-}) = j_{ECh} \tag{15}$$

In the given equation, T represents temperature, σ represents effective electric conductivities for the electrodes, and φ



represents phase potential for the electrodes. The symbols "+" and "-" indicate positive and negative electrodes. The variables \dot{q} , j_{ECh} , \dot{q}_{ECh} , C_p , and U denote the heat generation rate during battery operation, the volumetric current transfer rate, the electrochemical reaction heat due to electrochemical reactions, the heat capacity, and the open-circuit voltage of the battery, respectively.

The source terms, j_{ECh} and \dot{q}_{ECh} are obtained through equations that are dependent on the sub-model utilized, which in this case is NTGK (Newmann Tiedemann Gu and Kim) model. The NTGK model utilizes the parameters that are pre-defined as default in Ansys.

4.2 Model verification and validation for LHS

Numerical solutions are generally sensitive to the time step and element size of the meshed domain. The model is verified to evaluate the solution's dependence on the grid size and time step before performing the simulation. Figures 4a,b illustrate the examination of



four unique time steps (0.1 s, 0.25 s, 0.5 s, and 1 s) and four various grid sizes (40,000, 62,500, 90,000, and 122,250). The model with 90,000 elements and time steps of 0.25 s and 0.1 s exhibit comparable variations in the liquid fraction. The results show no significant variation with an increase in the number of elements to 122,250. However, the numerical algorithm diverges when a time step of 1 s is used. Therefore, a model with 90,000 elements and a time step of 0.25 s is selected for validating the model.

The accuracy of enthalpy-porosity (E-P) numerical methodology used in the present study is verified by comparing the results against the numerical and experimental results of Yadav and Sameer for melting of PCM with uniform heat flux as illustrated in Figure 5. The discrepancy between the experimental and numerical results can be attributed to the perfect insulation assumption made in the numerical model. It should be noted that achieving perfect insulation around the PCM domain is challenging in practical experiments, resulting in heat losses that are not accounted for in the numerical model.

4.3 Thermal performance comparison

The thermal cell is exposed to four uniform heat fluxes at the outer circumference for charging operation. The thermal cell is considered to be fully charged when the liquid fraction of PCM equals to one. Figure 6 illustrates the temporal variation of liquid fraction during the charging of silicon-based thermal cells. It is observed that the charging duration of the thermal cell is linearly proportional to the magnitude of the uniform flux. With an increase in flux from 625 W/m² to 5,000 W/m², the charging duration decreases by 89%. Figure 7 represents the temporal variation of the average temperature of PCM (silicon) as a function of input heat flux during charging. With increase in input heat flux from 1250 W/m^2 to 5,000 W/m² (400%), the average temperature of PCM increases by 77.7%.

The Li-ion battery was charged using a constant current charging method. The battery C-rate measures the current at which the battery is charged or discharged. A battery's C-rate measures how quickly it can be charged or discharged. A battery's capacity is often measured in units of 1C, which indicates that a fully charged battery with a rating of 3Ah should be able to supply 3A for 1 h. The charging duration of Li-ion battery decreases linearly with an increase in C rating as shown in Figure 8. The charging time at 0.5C current rating takes 110 min to charge the li-ion battery. For 1C current rating, the charging time was reduced by 48.82%-56.29 min. Similarly, as the C rating increases from 1.0C to 1.5C and 1.5C-2.0C, the charging time reduces by 33.38% and 24.98%, respectively. Overall, with an increase in C rating from 0.5C to 2C, the charging time decreases by 74.43%. Figure 9 shows the temporal variation of charged capacity at different C-rate. The obtained results indicate that a higher C-rating allows a shorter charging time.

Table 3 compares the charging duration and energy density of the thermal cell with Li-ion cell. Both gravimetric and volumetric densities are equally critical to design a compact LHS system. A significantly faster rate of charging is observed for silicon-based thermal cell compared to Li-ion cell except for the lowest heat flow of 7.693 W ($q'' = 625 \text{ W/m}^2$), the charging duration of thermal cell is 13.5% lower than the charging duration of Li-ion cell.





4.4 Economic performance

The economic performance of the HT-LHS system is estimated in terms of the Levelized cost of electricity (LCoE) as the financial indicator. The following assumptions are considered for the estimation of the economic performance:

- 1. 50 MW of spilled electrical energy to be stored using HT-LHS
- 2. 4 h of energy storage duration are considered
- 3. One discharge cycle per day is considered
- 4. Cost of spilled electricity is not included in the economic analysis

4.4.1 Levelized cost of electricity

The Levelized cost of electricity (LCoE) refers to the price point at which technology will generate enough revenue to cover





costs during its useful lifespan. The final result is a cost per kilowatt-hour that considers all relevant factors and expenses over the project lifetime. LCoE can be viewed as the annualized minimum price at which energy must be sold to break even the project's cost over its useful life. The LCoE analysis is a technique that helps to estimate the advantages and disadvantages of various energy systems. A lower LCoE indicates suggests that the method used to generate electricity is more economically viable. Equation 16 and Equation 17 are utilized to estimate the LCOE.

$$LCoE = \sum_{j=1}^{n} \left(\frac{\text{Total annual cost for } j^{ih} \text{ year}}{\text{Net annual electricity delivered in jth year}} \times \frac{1}{(1+d)^{j}} \right) \times CRF$$
(16)

Thermal cell				Li-ion cell		
Heat flow (W)	Charging time (minutes)			Charging time (minutes)		
7.693		142.76	165			
15.386		71.08				
30.772		35.33				
61.544		17.675				
Thermal cell			Li-ion cell			
Gravimetric energy density (kJ/kg)		3240.4		810.00		
Volumetric energy density (MJ/m3)		8,327.82		2,218.68		

TABLE 3 Comparison between thermal cell and Li-ion cell.

TABLE 4 Cost contributors of the LHS based thermal battery system.

Sl. No.	Cost contributor	Cost. (Rs)	Total cost (million Rs)
1	PCM (Silicon)	143/kg (Datas et al., 2016)	102.315
2	Shell material (Silicon carbide)	163.1/kg	0.238
3	Inconel 600 material	2969.46/kg	61.631
5	Overhead cost	15% storage tank cost	24.94
6	Indirect cost	30% of direct cost	57.383
7	Insulation	2% of storage tank cost	3.32
8	Supercritical CO ₂ cycle		3,324.016
Total capital cost	·	·	Rs.3573.84 million

$$LCoE = \sum_{j=1}^{n} \left(\frac{UCE_j}{(1+d)^j} \right) \times CRF$$
(17)

Where $UCE_j = Unit \text{ cost of electricity in the } jth \text{ year, } d = discount rate, CRF = Capital recovery factor.}$

CRF can be expressed as Equation 18.

$$CRF = \frac{d(1+d)^{n}}{(1+d)^{n}-1}$$
(18)

The total annual cost for the *j*th year and net annual electricity delivered in *j*th year depends on the type of Energy Storage system. The LCoE is a function of these two inputs, i.e., the yearly total cost and net annual electricity delivered. For high-temperature thermal battery energy storage system, the estimation for these two inputs (the total annual cost and net annual electricity delivered) is explained in the following section.

For an energy storage system, the total annual cost includes the annual cost of the principal amount of the loan, the annual cost of interest on the loan, annual cost of return on equity, annual costs of the operation and maintenance, and annual costs associated with the renewal of the system. The life cycle cost is the sum of yearly costs associated with the project. The costs associated with a particular type of energy system can be summarized as follows:

- I. Capital Cost
- II. Annual cost of the principal amount of the loan
- III. Annual cost of interest on the loan
- IV. Annual cost of return on equity
- V. Annual costs of the operation and maintenance of the system
- VI. Costs associated with the renewal of the system
- VII. Other expenses

The input parameters considered to estimate LCoE are capital cost (C_o), annual operation and maintenance costs (C_j), useful life of the project (n), discount rate (d), and salvage value of project (s).

4.4.2 LCoE for thermal battery system

As described in preceding Section 2, high-temperature latent heat storage (HT-LHS) system having silicon as PCM is considered as thermal battery in this study. The Capital expenditure for a single tank LHS is the summation of direct and indirect costs. The direct cost includes storage material cost, storage tank cost, HTF cost, and overhead cost. The miscellaneous expenses such as piping, valves, fitting, and electrical costs are considered part of the overhead cost. The overhead cost is considered as 15% of the entire cost associated with the storage tank. The indirect cost covers the sales tax, fabrication, and contingency costs. The indirect cost is considered 30% of the direct cost of the system (Nahar, 2002). The cost contributors that are supposed to evaluate the system cost are listed in Table 4. The total capital cost was TABLE 5 Financial parameters for LCoE calculation.

Capital cost	As per calculation		
Debt-Equity ratio (%)	70-30		
Return on equity (%)	20 for the first 10 years and 24 from 11th year onwards		
Loan repayment period (years)	12		
Interest rate on loan (%)	9.67		
Useful life (years)	25		
Discount rate (%)	8.61		
Depreciation (%)	4.67 for first 15 years, 2 from 16th year onwards		
Working capital	One month of O&M, 15% spares; 2 months of receivables		
Interest in working capital (%)	11.17		
O&M expenses (%)	1.5		
O&M expenses (%)	3.84		
Levelized tariff rate	Rs 12.08/unit		



computed for a thermal battery integrated sCO_2 cycle system with a power rating of 50 MW and an energy capacity rating of 200 MWh, comparable to 4 hours of storage. The capital cost is brought to the present value by considering the time value of money. The estimated capital cost of the integrated thermal battery system is estimated to be Rs 3753.863 million.

Cost of operation and maintenance (C_j) : The operation and maintenance (O&M) is the annual cost associated with the routine supervision and services performed annually to ensure the smooth functioning of the HT-LHS system. The escalation in operation and maintenance costs are considered to cater to the wear and tear during operation.

Discharged Energy ($E_{discharge}$): This is the net annual energy discharged from the HT-LHS system. It depends on cyclic degradation, the number of cycles, and round-trip efficiency. Cyclic degradation is not considered as the life cycle of LHS is 25 years. One discharge cycle per day and 95% round-trip efficiency are considered for the LHS storage.

Salvage value: It is defined as the monetary worth of the project after the end of its useful life. In the context of LHS projects, it is the economic value of the controls and instrumentation at the end of their useful life. This control and instrumentation have the potential to be utilized in other services, which will result in a net reduction in investment costs.

Zero salvage value is considered for the calculation and can be estimated as Equation 19.

Present salvage value =
$$\frac{s}{(1+d)^n}$$
 (19)

Financial parameters considered to calculate the LCoE of the LHS system integrated with the sCO_2 cycle are mentioned in Table 5; (Shri Gireesh, 2016).

LCoE for HT-LHS system (Thermal battery) integrated with sCO_2 cycle with a rated power of 50 MW and an energy capacity of 200 MWh is found to be 9.547 Rs/kWh.

4.5 Sensitivity analysis

A sensitivity analysis is conducted to estimate the degree of reactiveness of LCoE with individual financial parameters. In the present study, the sensitivity of LCoE is represented as a function of discount rate (DR), loan repayment years (LRY), the interest rate on loan (IOL), return on equity (ROE), and operation and maintenance cost (O&M). Figure 10 illustrates the sensitivity of LCoE with the variation of financial parameters from -40% to +40% for thermal and Li-ion batteries, respectively. From the sensitivity plot, LCoE is most sensitive to ROE and least sensitive to DR for thermal battery. IOL is the second most sensitive parameter having a significant effect on LCoE. However, LRY has the opposite effect on LCoE as compared to other financial parameters. Unlike other parameters, LCoE decreases by 9.23% with a variation of LRY from -40% to +40%. For Li-ion battery, ROE and LRY are the most and least sensitive parameters, respectively.

5 Conclusion

In conclusion, the thermo-economic study presents compelling evidence regarding the promising potential of a metallic hightemperature latent heat storage (HT-LHS) system integrated with sCO₂ recompression cycles for simultaneous heat and power generation.

Firstly, silicon as PCM in HT-LHS systems showcases notable advantages, including significantly shorter charging durations compared to equivalent Li-ion batteries. This reduction in charging time, coupled with the approximately fourfold higher energy density of thermal cells compared to Li-ion cells, underscores the efficacy of HT-LHS technology in energy storage applications. A comprehensive economic analysis of the integrated HT-LHS system and sCO₂ recompression cycle reveals a reasonable LCoE of 9.547 Rs/kWh for 200 MWh storage capacity. Sensitivity analysis further highlights the robustness of HT-LHS technology, particularly regarding return on equity (ROE), thus affirming its economic viability. However, further financial analyses of Li-ion battery are warranted to compare the two technologies comprehensively. Thus, our study underscores the importance of thorough assessments and continued research to unlock the full potential of HT-LHS systems in the evolving landscape of energy storage technologies.

Data availability statement

The raw data supporting the conclusions of this article will be made available by the authors, without undue reservation.

Author contributions

AR: Writing – original draft, Writing – review and editing, Conceptualization, Data curation, Formal Analysis, Investigation, Methodology, Validation, Visualization. SV: Conceptualization, Data curation, Formal Analysis, Investigation, Methodology, Validation, Visualization, Writing – original draft, Writing – review and editing. DR: Writing – original draft, Writing – review and editing, Funding acquisition, Project administration, Resources, Software, Supervision. RK: Funding acquisition, Project administration, Resources, Supervision, Writing – original draft, Writing – review and editing. HG: Funding acquisition, Resources, Supervision, Validation, Writing – original draft, Writing – review and editing.

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Conflict of interest

The authors declare that the research was conducted in the absence of any commercial or financial relationships that could be construed as a potential conflict of interest.

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Nomenclature

BESS	Battery energy storage system	TIPV	The	ermionic photovoltaic
CCT	Climate change technology	UCE	Uni	it cost of electricity
CRF	Capital recovery factor			
CSP	Concentrated solar plant	Symbols		
DOD	Depth of discharge	-	a	
DR	Discount rate	C Charging rate		ging rate
HT	High-temperature	C _o Capital cost		al cost
IOL	Interest rate on loan	Cj	C _j Operation and maintenance cost	
LCoE	Levelized cost of electricity	c _p Specific heat at constant pressure		fic heat at constant pressure
LHS	Latent heat storage	d	Disco	ount rate
LRY	Loan repayment year	h _{sl}	Laten	t heat of fusion
MSMD	Multi scale multi domain		Volur	metric current transfer rate
NTGK	Newmann Tiedemann Gu and Kim	q _{ECH}	Electr	ochemical reaction heat
TCHS	Thermochemical heat storage	s Salvage value		
TES	Thermal energy storage	sCO ₂	sCO ₂ Supercritical CO ₂	
O&M	Operation and maintenance	Greek Symbols		
РСМ	Phase change material	β	Ν	Melting fraction
PV	Photovoltaic	e	S	Small constant
PVC	Present value cost	$\rho~(kg/m^3)$		Density
ROE	Return on equity	μ (Pa.s)	Ι	Dynamic viscosity
SD	Self discharge	Φ (V)	F	Phase potential
SHS	Sensible heat storage	σ (S/m)	E	Electric conductivity